

# ODYSSEE

## Machine Learning (ML) based ROM for Real-Time Optimization With applications for engineering (**and everything else**)

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**CADLM**  
TRANSFORMING DATA INTO INTELLIGENCE

Analytics for Industry  
Products, Methods, Processes

- Data Fusion and Machine Learning
- Predictive real-time modelling
- Data Mining and Modelling
- Optimization and Robustness

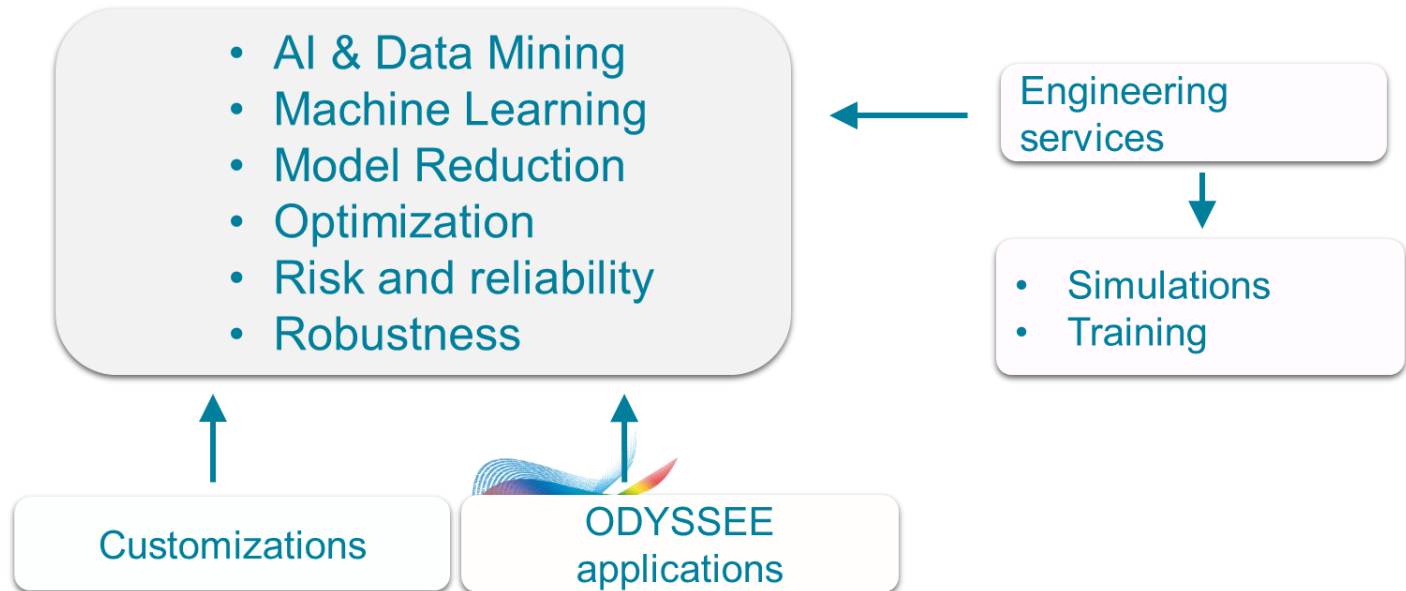
**ODYSSEE**  
Explore new industrial horizons

“CADLM accelerates product design and development via **real-time** parametric simulations with our optimization, machine learning and AI tools”



## **ODYSSEE - Explore new industrial horizons**

Optimal Decision Support System for Engineering and Expertise



# Our customers



THALES



BOMBARDIER



Toyota Motor North America



TOYOTA



OPEL



azbil

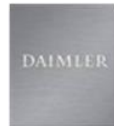


ULVAC



TOYOTA BOSHOKU

AsahiKASEI



Panasonic

Toyoda Iron Works



mazda



# Which came first: Data or Computing ?



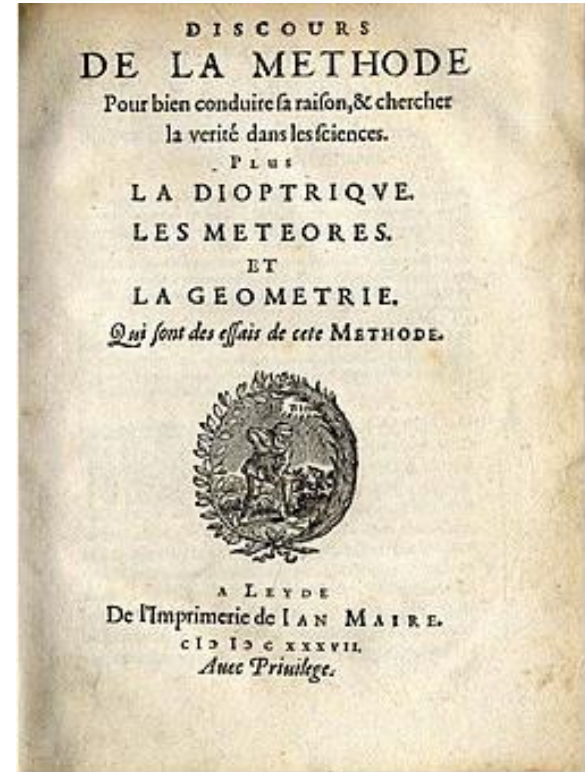
ODYSSEE

Explore new industrial horizons

Machine Learning (ML) with CAE will greatly impact the product development process.

Focus Areas:

- Creating **Predictive Models** with Machine Learning
  - Avoid long, complex and costly (pre/solve/post) simulation process.
  - **Simulation can provide the training data**
- Support **Controller / Sensor Learning** providing additional training data
  - Will replace the costly analysis process.
  - Enable the creation of autonomous systems.



# Our solutions aim at COST reduction

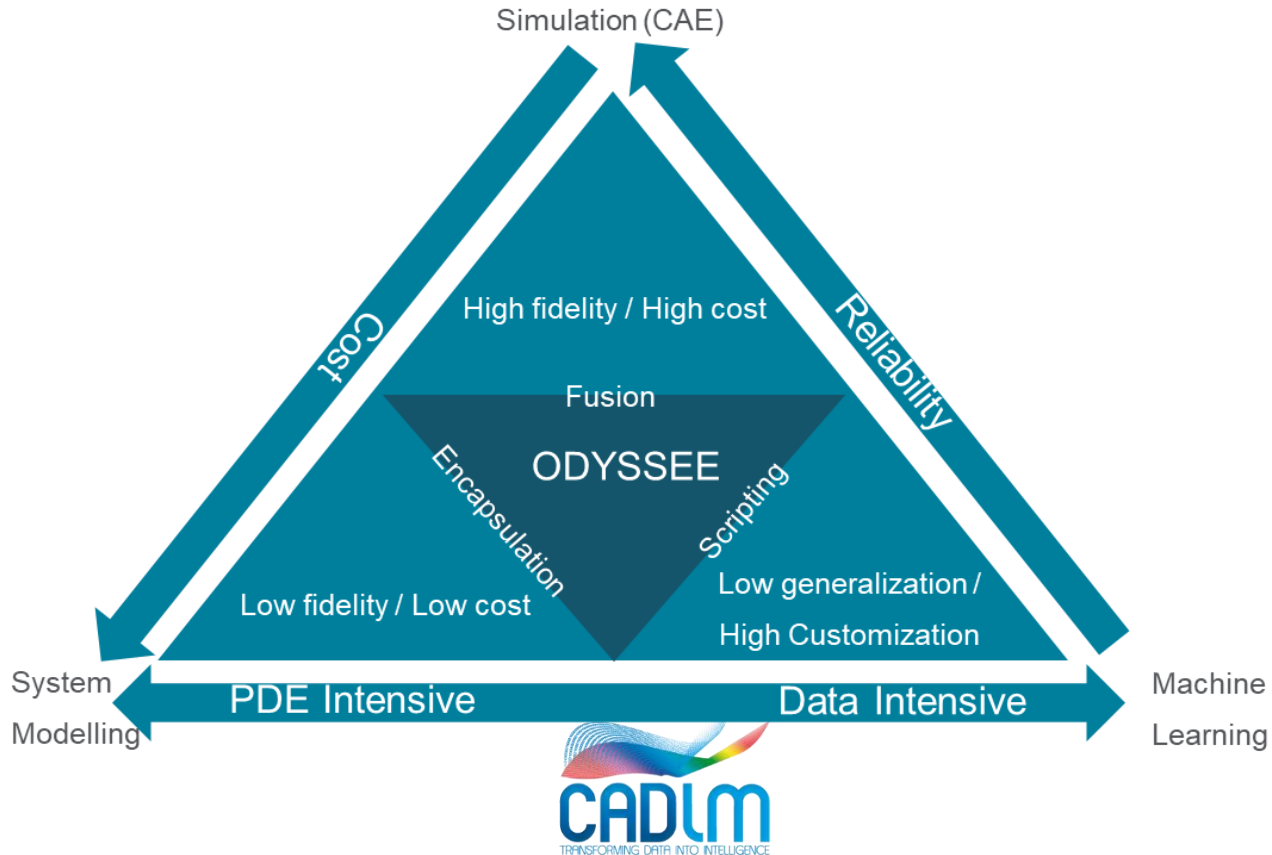
## Customers' problems (**COST**)

- **C**omputing (**HW/SW**, CPU, Energy, ...)
- **O**ptimization (**Iterations, curse of dimensionality**, precision of surrogate models,, parametric studies, stochastics)
- **S**imulation (**Model size and complexity**, Multi-physics, multi-scale, encapsulation, model transfer without loss of confidentiality)
- **T**ime (**Real-time**, pre/post automation, animations, etc.)

## Our Applications

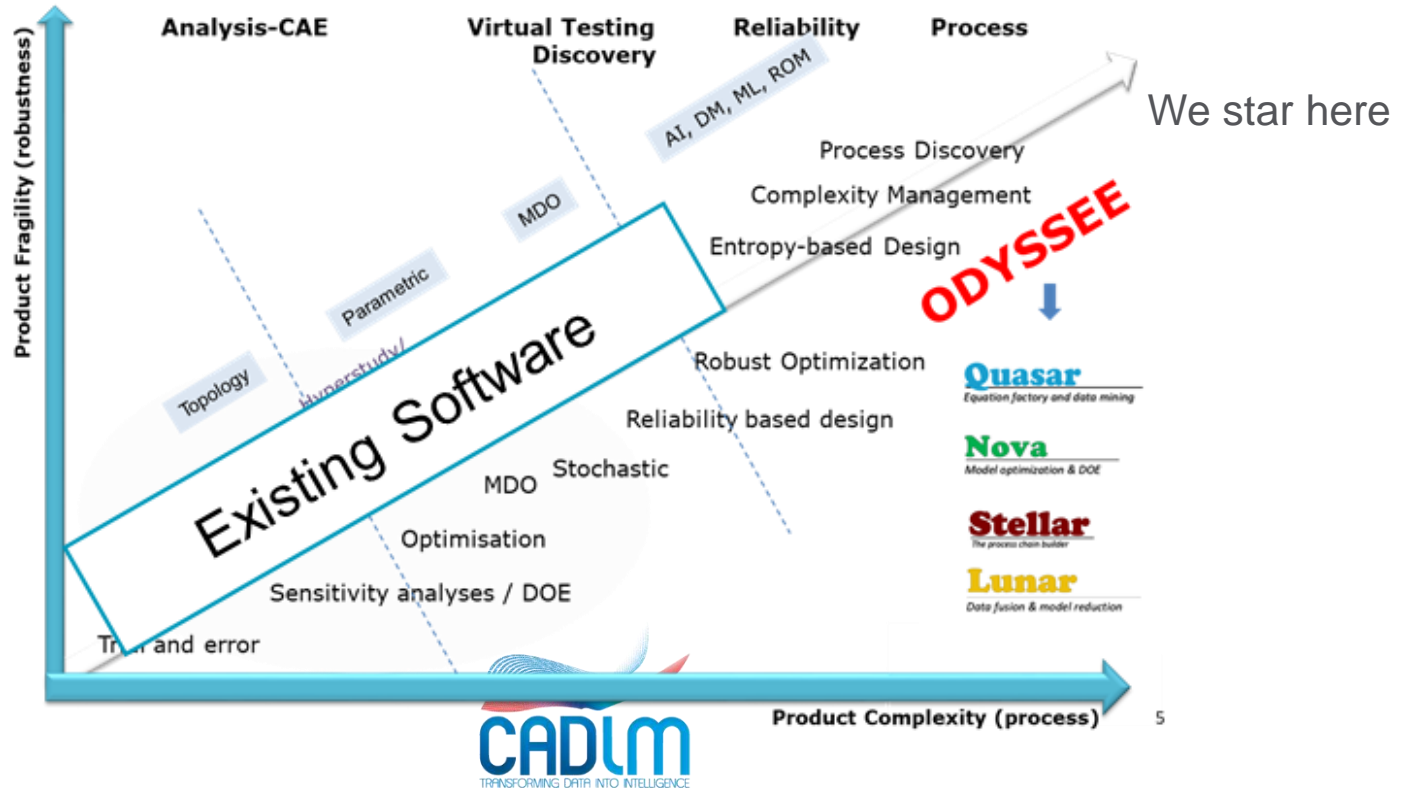
1. **Real-Time** predictive modeling and optimization (CAE or test data)
2. **Image , Sound & Sensors** compression, identification, learning, prediction
3. **Fault** prediction (Sensor data)

# Our Technology Positioning



# Our Added Value Raises with Design Complexity

Positioning



# ODYSSEE Full Package

## LUNAR

Model Reduction &  
Optimization

## QUASAR

Machine Learning

## NOVA

Optimization

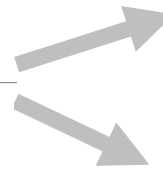
- QuasarOnline,  
PULSAR, ...

ODYSSEE is a modular package:



## Lunar

*Data fusion & model  
reduction*



## Quasar

*Equation factory and data  
mining*



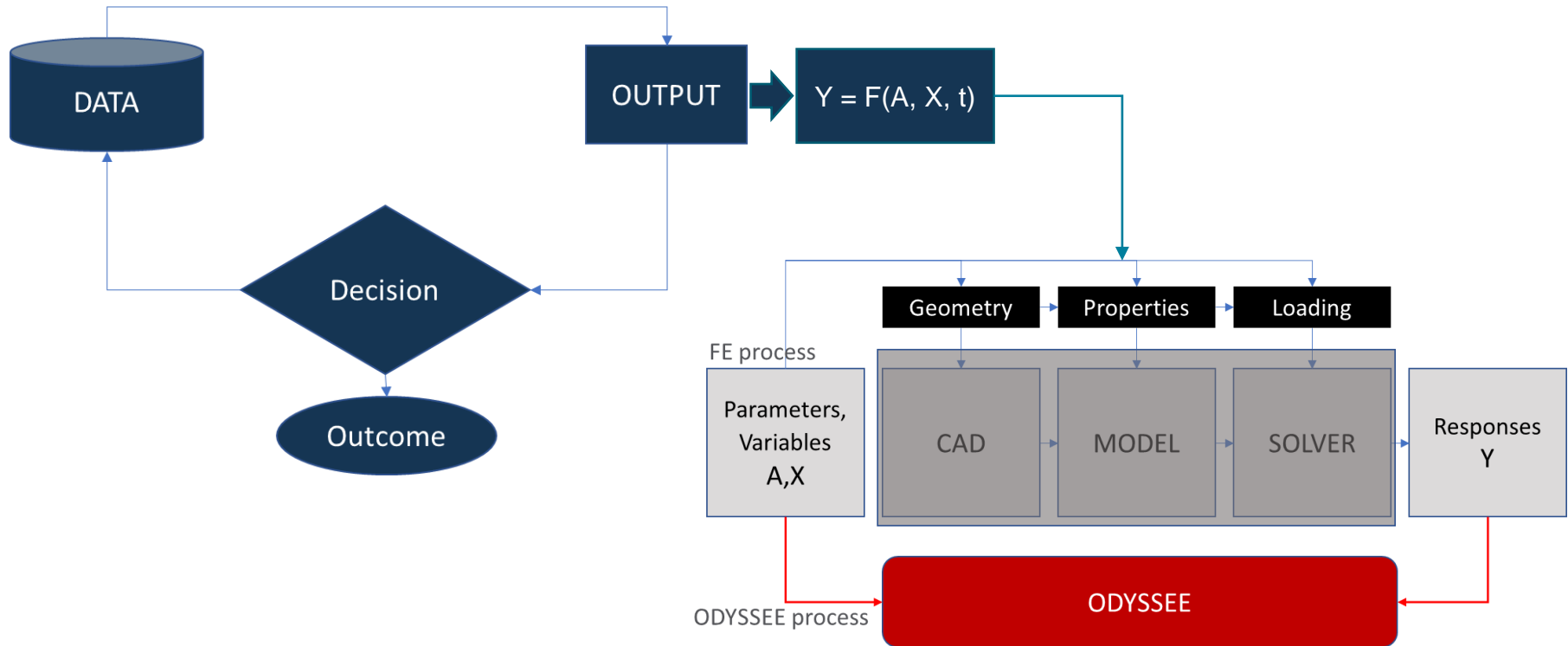
## Nova

*Model optimization &  
DOE*

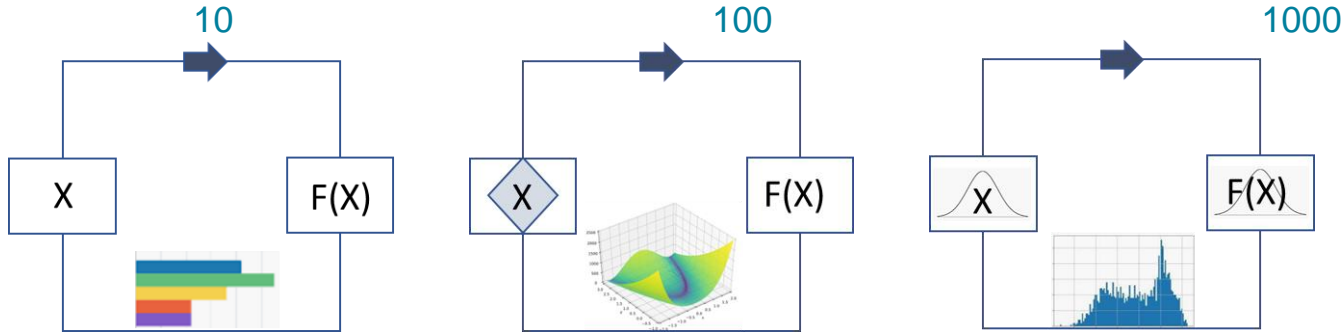




# Current Design process



# Why LUNAR?

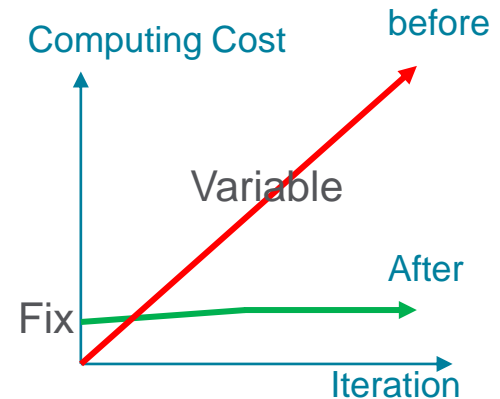


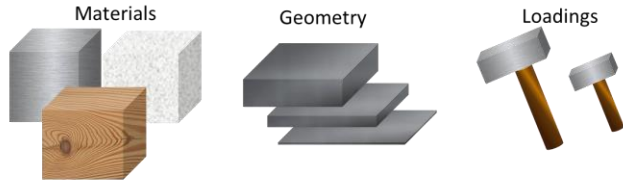
- Parametric Studies
- Trial and error

- Optimization
- Model Fitting

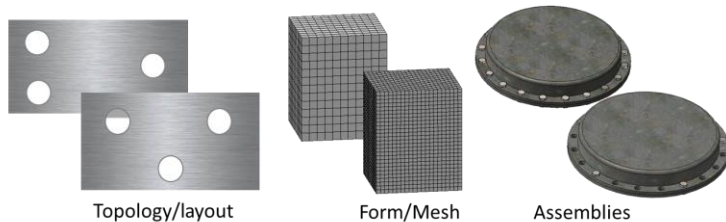
- Reliability Studies
- Robustness

**All require numerous solver runs**





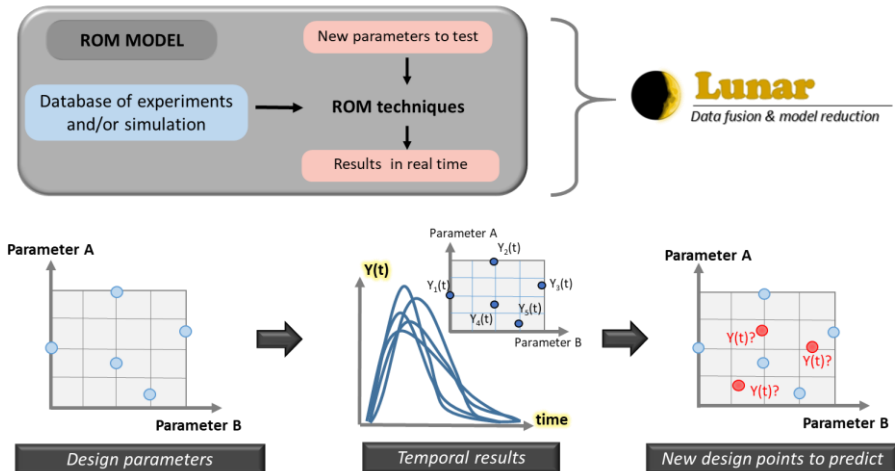
Model parameters may be simple or complex



Design Parameters

DOE based data collection

Model Order Reduction (ROM) + ML



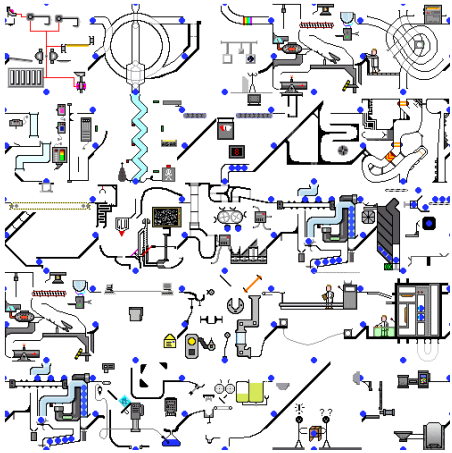


# Lunar

*Data fusion & model reduction*

## Reduced Order Modelling (ROM) and Real-Time Optimization

ODYSSEE (LUNAR/ QUASAR/ NOVA)

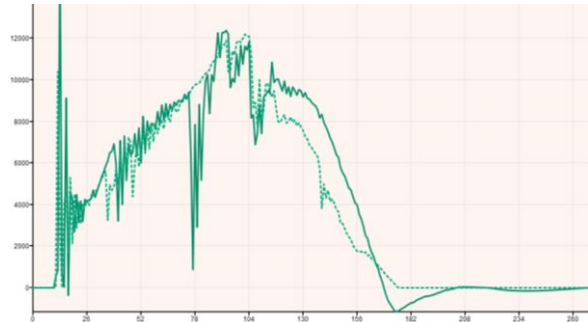


# Lunar exploits ROM

Lunar is based on Model Order Reduction techniques

May predict **temporal** & static responses

May predict bifurcation and non-linear response



*Prédiction des réponses temporelle*

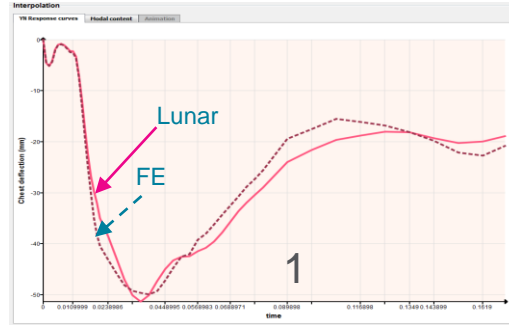
Note:

- **ROM is not a response surface method**

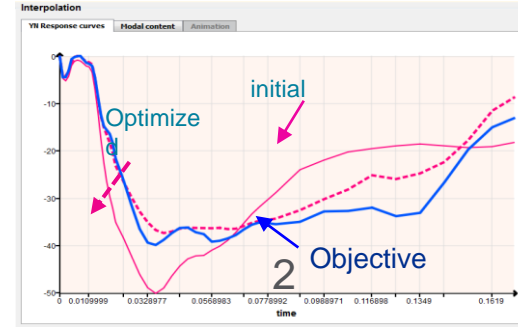
A response surface uses polynomial or other interpolation functions as « a-priori » applied to scalars (temporal responses are treated as scalars -> long and non repeatable: needs to be re-done for every prediction)

- **ROM predicts complete time responses and is based on physical modes of behavior**

Time responses in real time



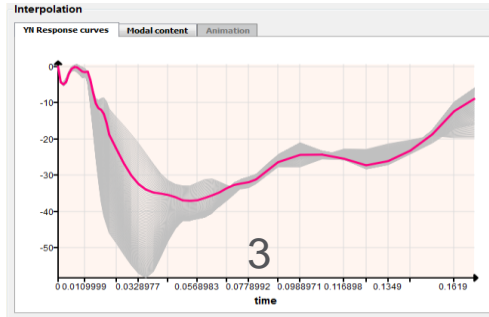
Optimization and parameter fitting



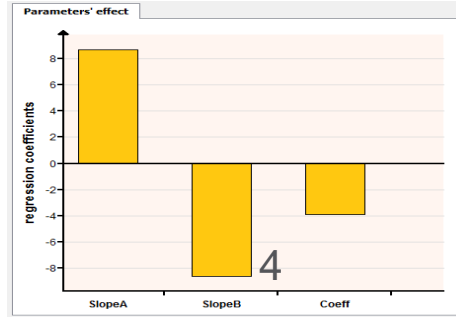
**Lunar**

Data fusion & model reduction

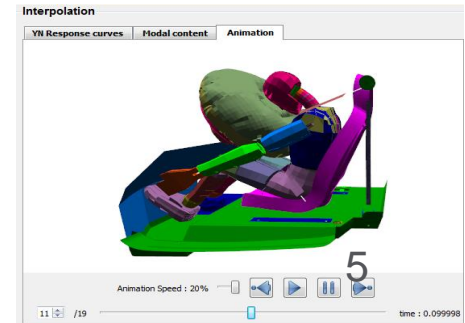
Population studies (corridors)



Sensitivity analysis of parameter effects



Real-time animations



## Real-time computing

- Zero-computing effort for parametric studies and optimization
- Corridors & Population generation (statistics)
- Parameter effects
- DOE's

## Software and physics independent

- Works with Structural, Thermal, CFD, Acoustics (Ls-dyna, Radioss, pam-crash, MSC Nastran, Marc, Adams, Cradle CFD, Actran)

## Automation/Parser

- Automatic post-preprocessing and rating
- Multi-channel

## Reduces CAE computing effort

- Allows for a few, wisely selected sampling points
- Adaptive learning that allows you to improve as you learn

## Precision & completeness

- Full time history output (not only scalars)
- Physical domain decomposition and not fitting (it is NOT a Response Surface Method!)

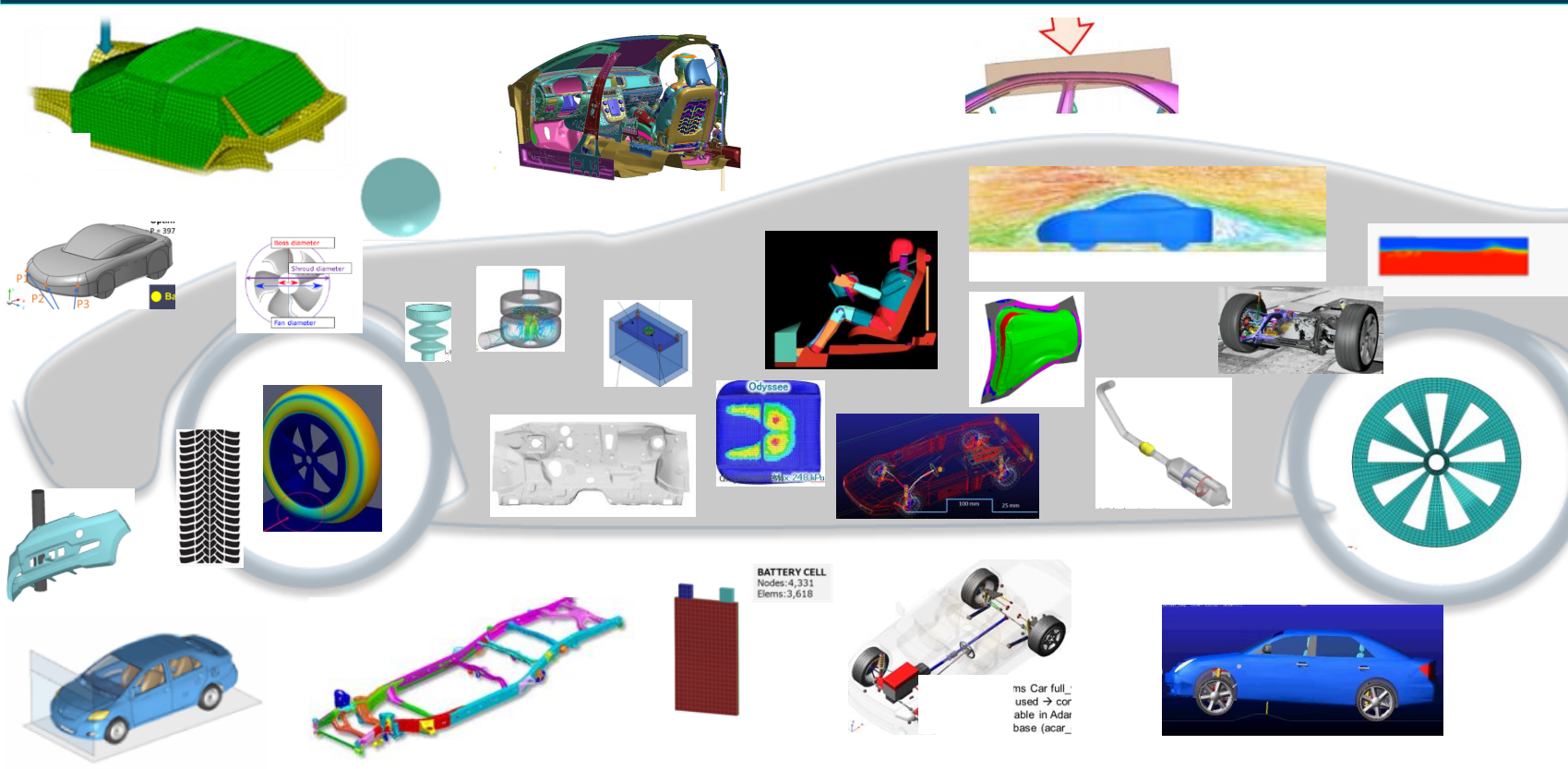
## Can produce 3D animations

- No interpolations but reconstructions
- Stress/displacement iso-value reconstruction

## Evaluation tools included

- Quality of parameters • Quality of DOE
- Best method for your application

# CADLM offer (AI/ML/ROM based optimization)

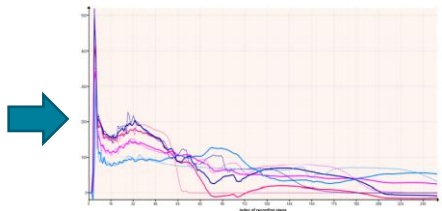
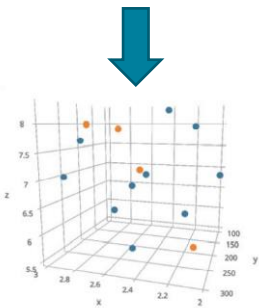
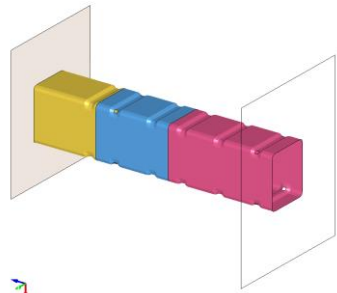


- Aerodynamics
- Aeroacoustics
- Battery
- Crash
- Electric
- Ergonomics
- Exhaust
- Fuel tank
- Heat
- Impact
- Noise
- Safety
- Stamping
- Strength
- Styling
- Tires
- Vibroacoustic

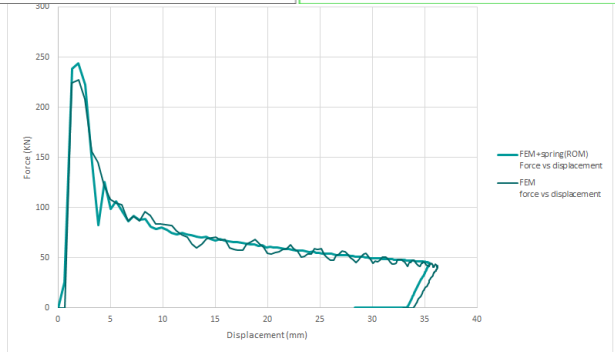
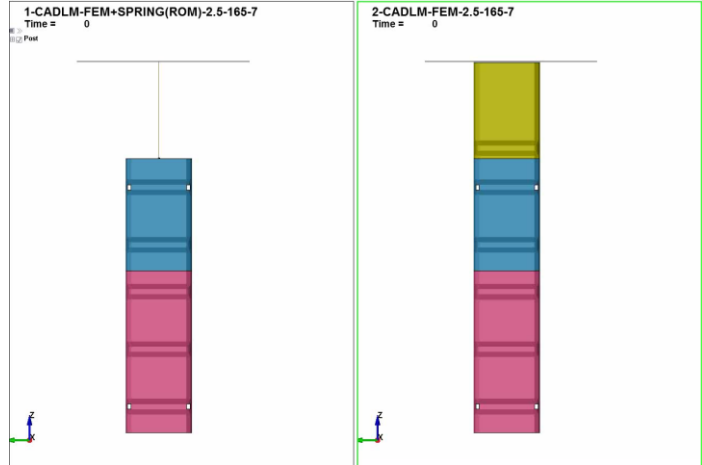


# Sub-system encapsulation (\*.k include)

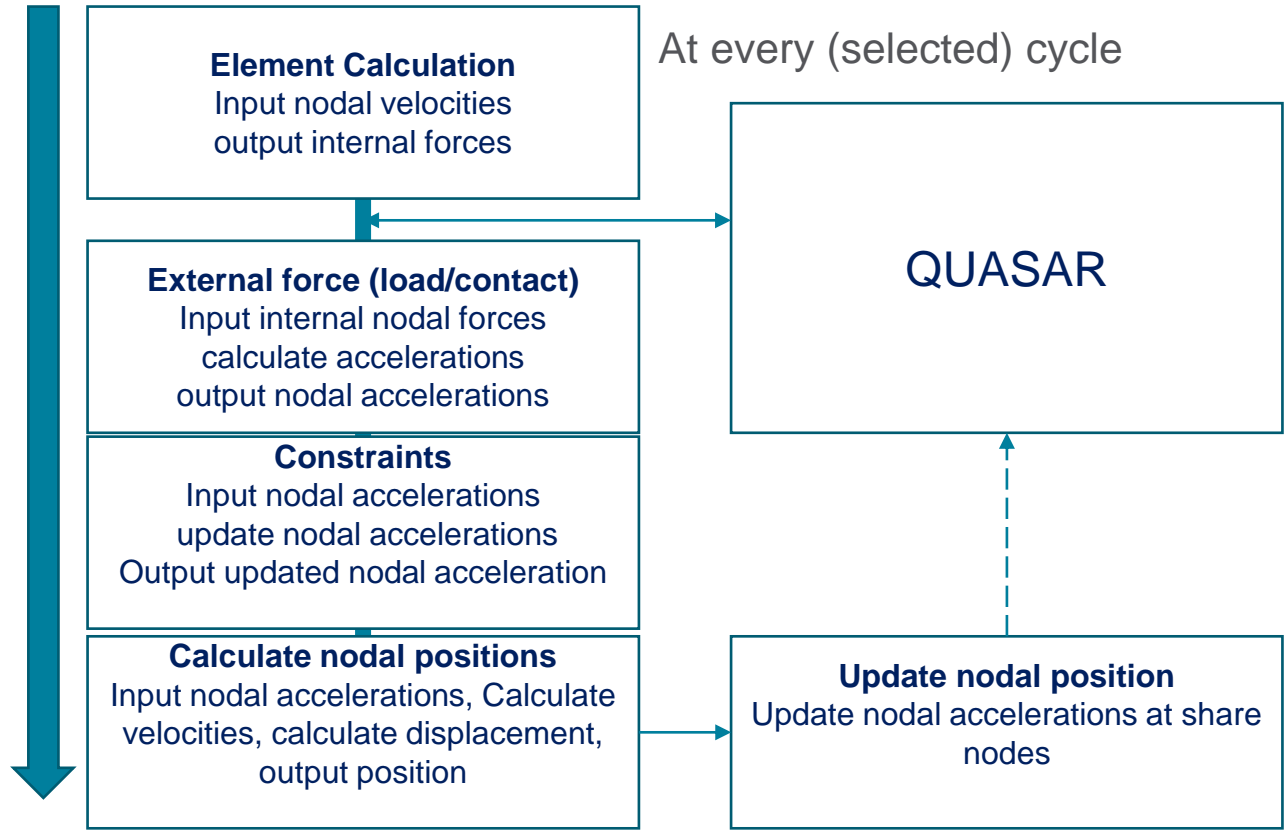
CADLM - global model - mm/ms/kg/GPa/KN/mm



thickness :2.33 mm; mass RW:300. kg; velocity RW:7.22 mm/ms



# Quasar-IsDyna coupling



**\*DEFINE\_QUASAR\_COUPLING** **\*DEFINE**

**\*DEFINE\_QUASAR\_COUPLING**

Purpose: Define LS-DYNA node/node set that interacts with Cadlm's QUASAR ROM model. Each coupling needs to have its own keyword card and will not accept multiple entries.

Card 1	1	2	3	4	5	6	7	8
Variable	NODE	TYPE	ROMID	PID	PTYPE	IOPT	CID	EX_ND
Type	I	I	I	I	I	I	I	I

Card 2	1	2	3	4	5	6	7	8
Variable	FILENAME1							
Type	A80							

Card 3	1	2	3	4	5	6	7	8
Variable	FILENAME2							
Type	A80							

Optional card for user defined constants. Eight floating point per card until reach next keyword card.

Card 4	1	2	3	4	5	6	7	8
Variable	Variables – ncards							
Type	F							



# \*QUASAR Keyword

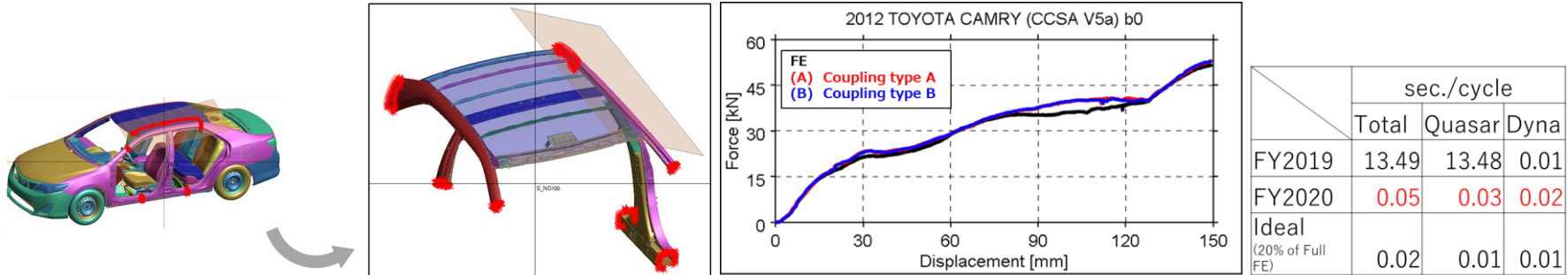
## DYNA-QUASAR coupling

- Motivation

- Want to reduce calculation time by utilizing ODYSSEE

1. Cut out only the parts you want to evaluate from the full car model
2. Boundary characteristics are predicted using ODYSSEE

- The calculation stability is greatly improved, and the calculation is possible to the end.
- The calculation speed was also greatly improved and could be reduced to 3 times the target value.

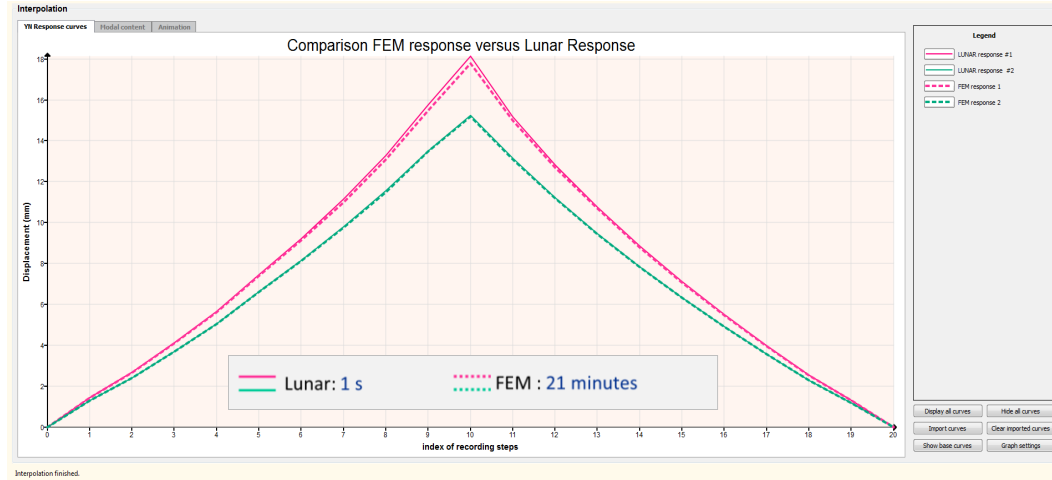
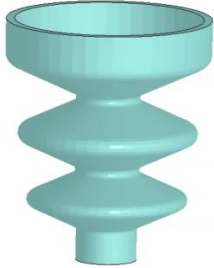


1

Courtesy of JSOL

# Rubber joint (Is-dyna) – Material Nonlinearity

CADLM - rubber compression  
Time = 0



**15 runs, 3 parameters**

Mass => X1

Mooney Rivlin parameters => X2, X3

**Output Channel**

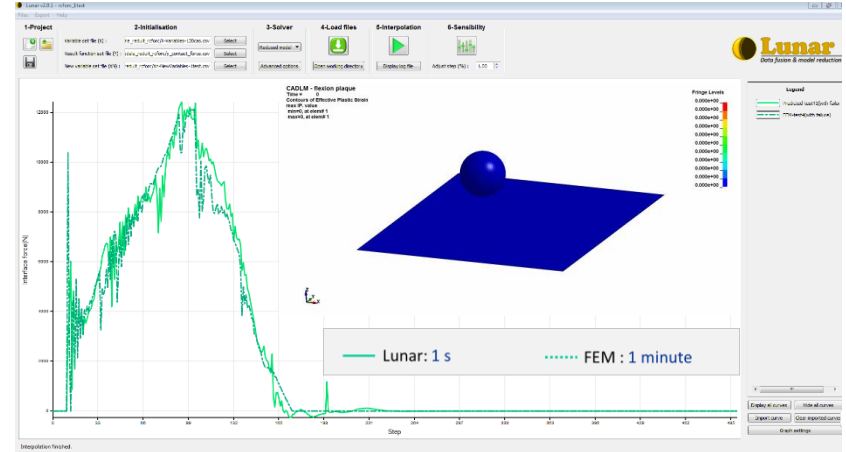
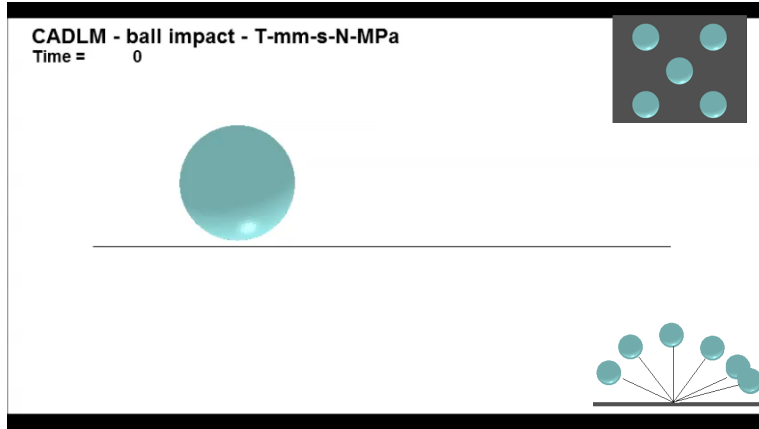
Displacement at extremity of joint =>  
Y1

**Elapsed Time**

FE = 21 minutes

LUNAR = 1 sec

# Ball impact with rupture (ls-dyna)



## 15 runs, 4 parameters

Position X,Y => X1, X2

Ball speed at impact => X3

Impact angle => X4

## Output Channel

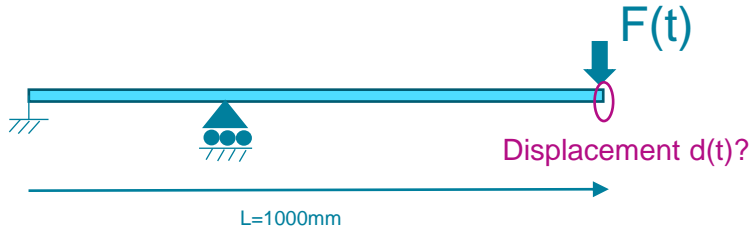
Contact force => Y1

## Elapsed Time

FE = 1 minute

LUNAR = 1 sec

# Plate flexion (Influence Lines)



## Influence Lines

### 2 model parameters

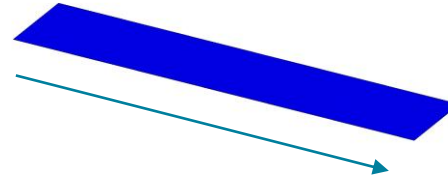
Plate thickness  $\Rightarrow X1$

Mobile support distance  $\Rightarrow X2$

### Output channels:

displacement-time over the complete spatial domain  $\Rightarrow Y1$

CADLM - flexion planche - X1:epaisseur=35mm - X2:Distance appui mobile=4  
Time = 0  
Contours of Effective Stress (v-m)  
max IP value  
min@0, et elem# 1  
max@0, et elem# 1



179.24 frame/sec

**15 DOE  
runs**

Finite Elements

Lunar

1 minutes / simulation

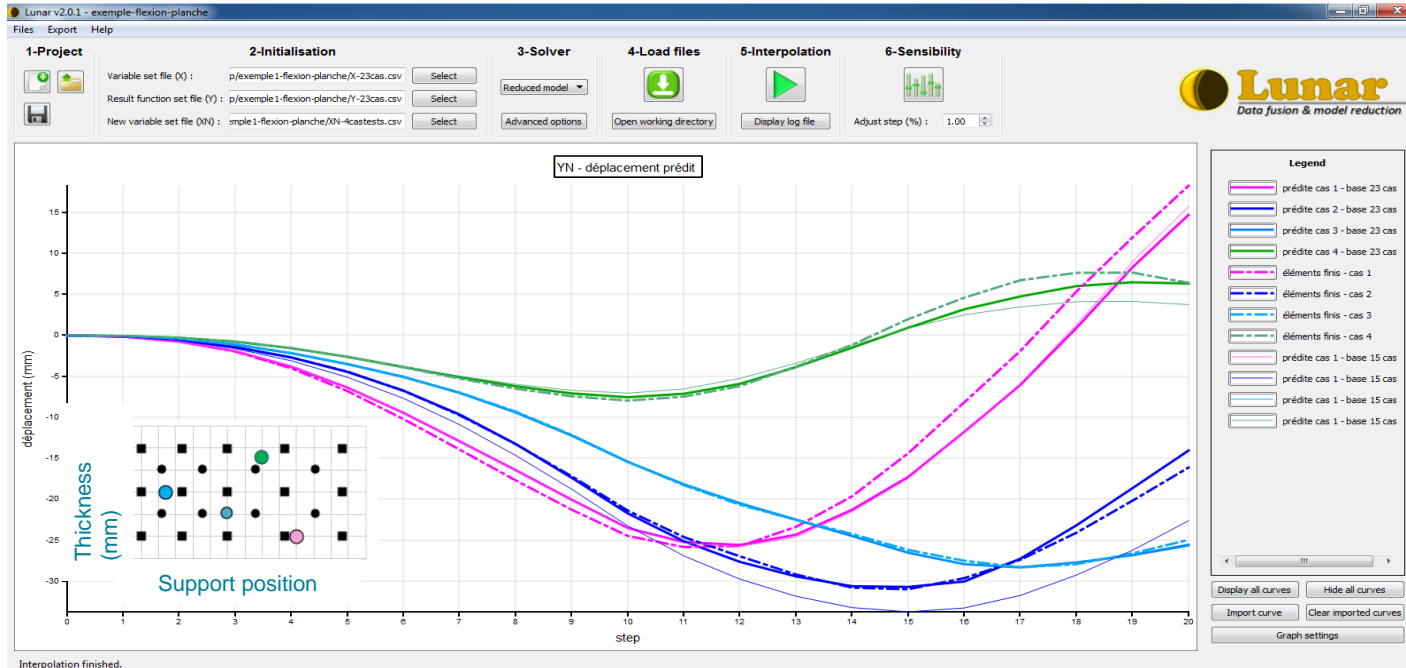
1 second

# Plate flexion (Influence Lines)



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Lunar: 1 s    Lunar: 1 s    FEM : 1 minutes  
DOE    DOE    DOE



# Example – Rubber Compression Joint

CADLM - rubber compression  
Time = 0



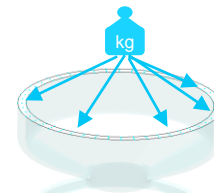
## 3 model parameters

Mass => X1

Mooney Rivlin parameters (A, B) =>  
X2, X3

## Output channels:

Displacement at extremity of joint => Y1



**13 DOE runs**

Finite Elements

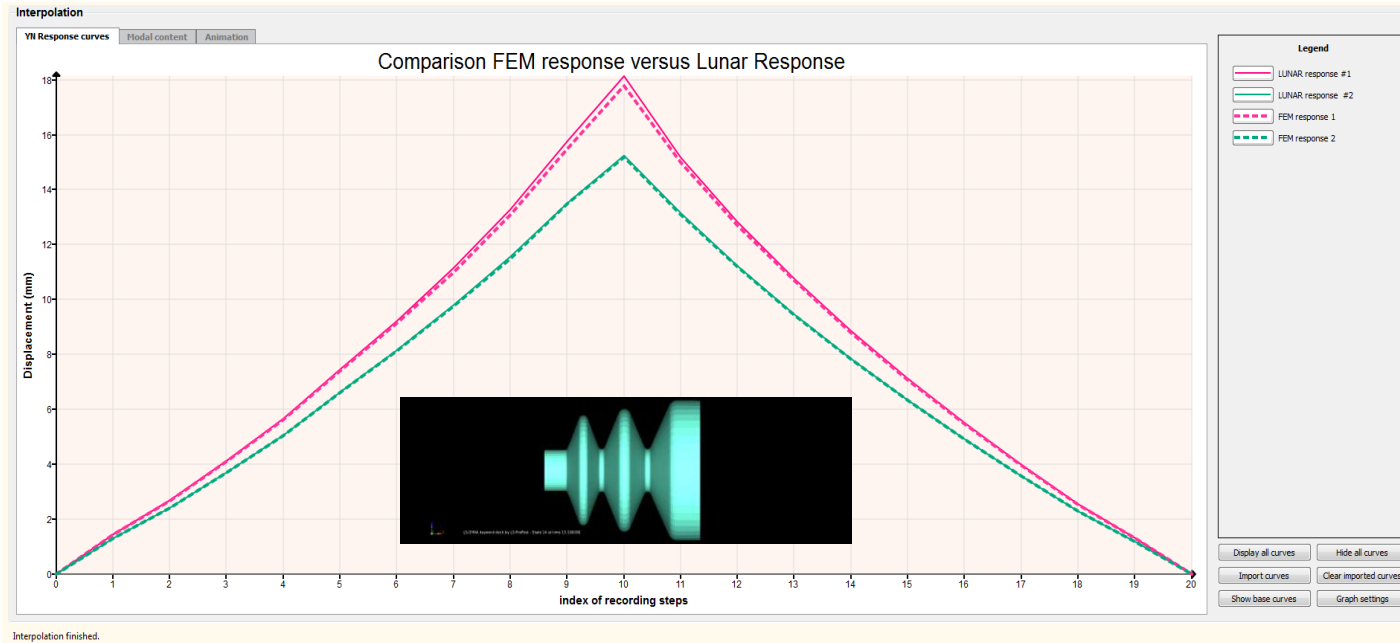
Lunar

21 minutes /  
simulation

1 second



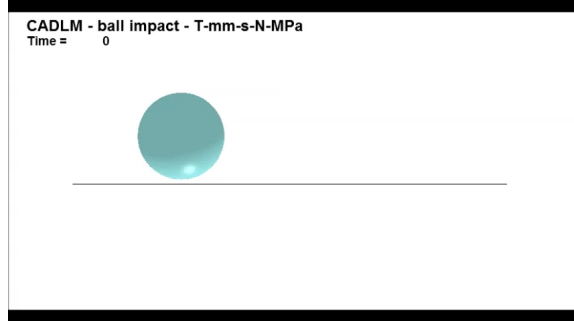
# Rubber Joint Compression



Lunar: 1 s

FEM : 21 minutes

# Example: Ball impact on plate



## 4 Model parameters:

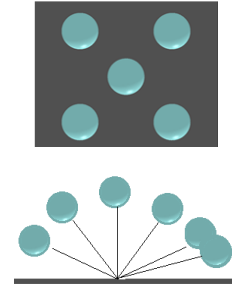
Position  $X, Y \Rightarrow X1, X2$

Ball speed at impact  $\Rightarrow X3$

Impact angle  $\Rightarrow X4$

## Output Channel:

Contact force  $\Rightarrow Y1$



**15 DOE runs**

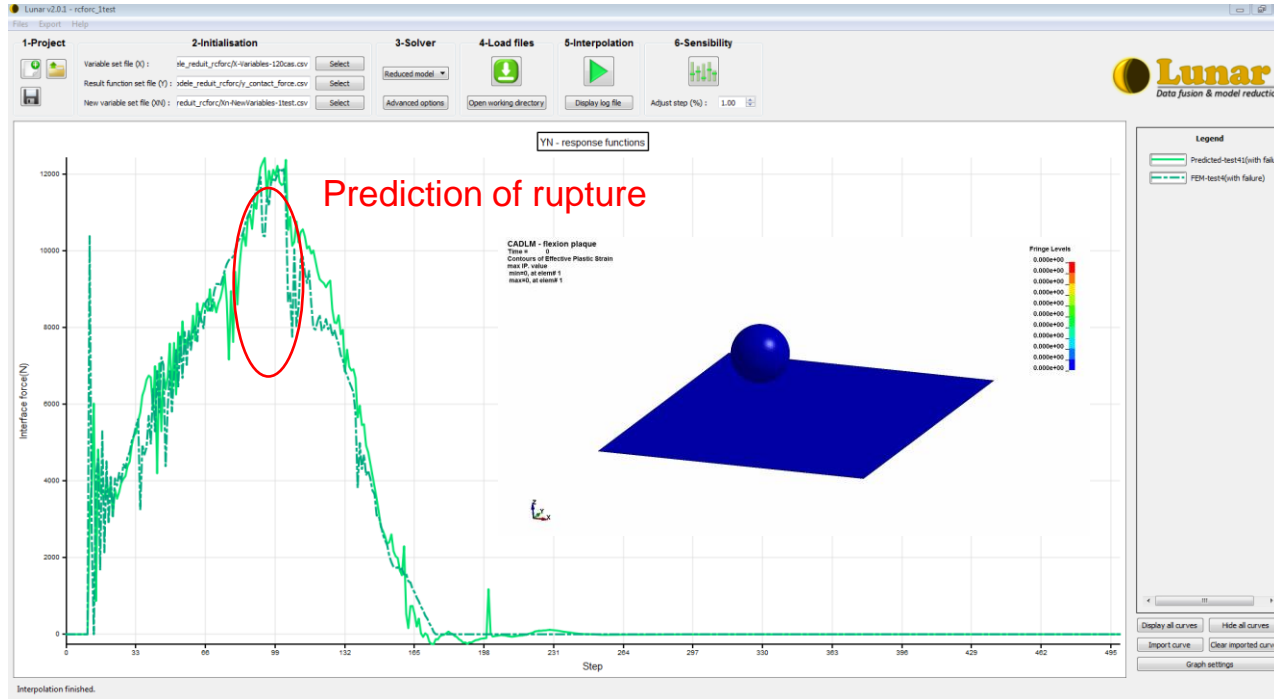
Finite Elements

Lunar

1 minute / simulation

1 second

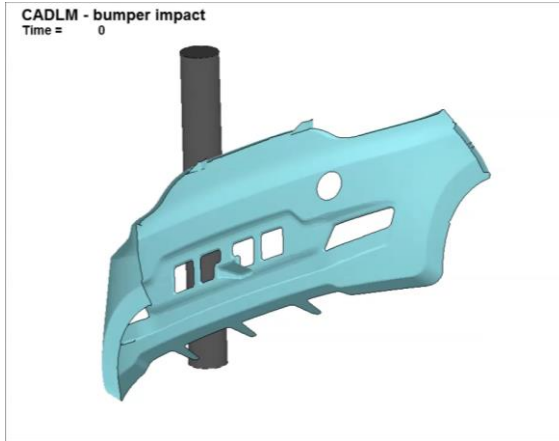
# Ball impact on plate



— Lunar: 1 s

..... FEM : 1 minute

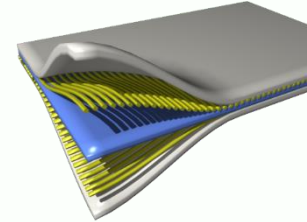
# Pole impact of composite bumper panel



## Model parameters:

Fiber orientations in:

- Layer 1: X1
- Layer 2: X2
- Layer 3: X3
- Layer 4: X4



## output channel:

Y : Bumper deceleration

**15 DOE runs**

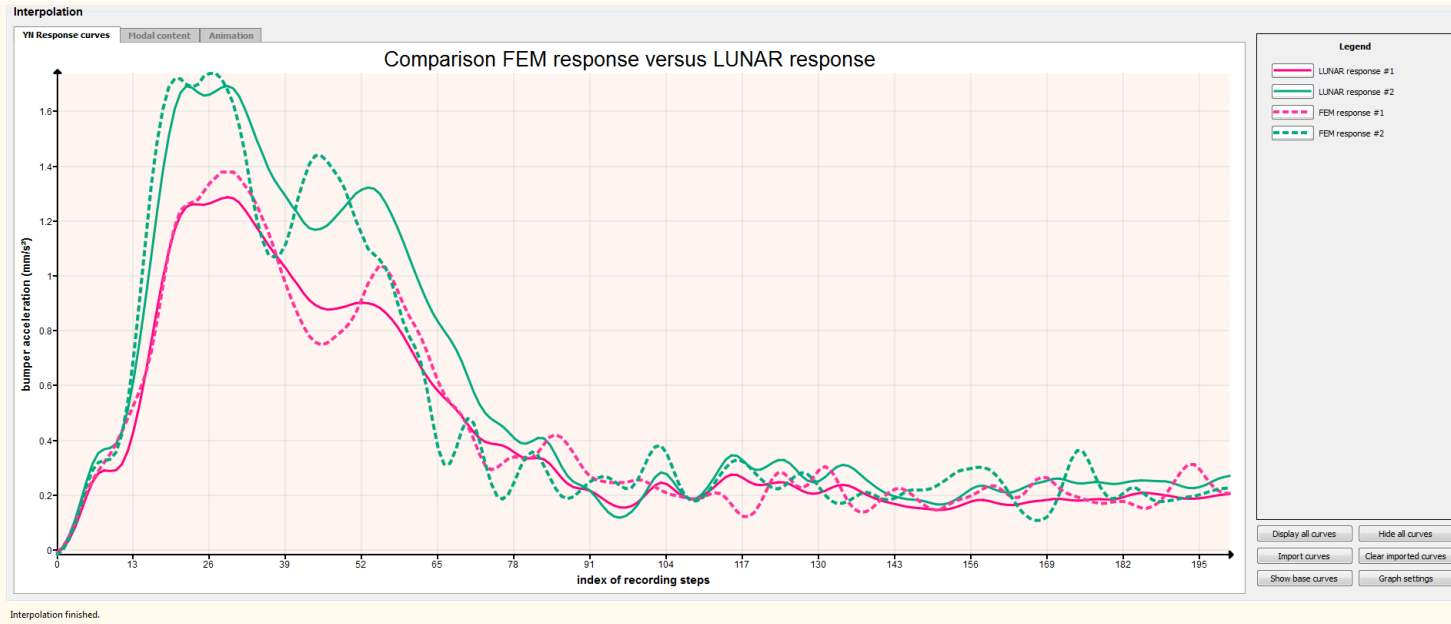
Finite Elements

Lunar

6 hours/ simulation

1 second

# Crash composite bumper panel



— Lunar: 1 s

⋯ FEM : 6 hours

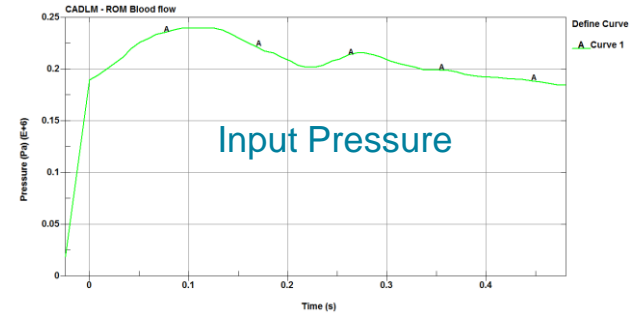
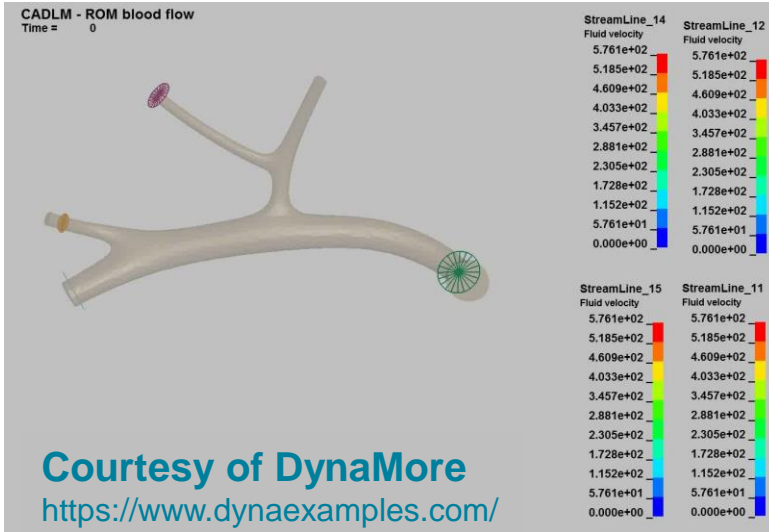
# Example - Fluid blood flow model

## 2 Model parameters:

Blood pressure injected in input surface => X1  
Blood pressure in one output surface => X2

## Output channels:

Flow rate measured at the input/output surfaces



## DOE 8 runs

Finite Elements

Lunar

2.5 hours per  
simulation

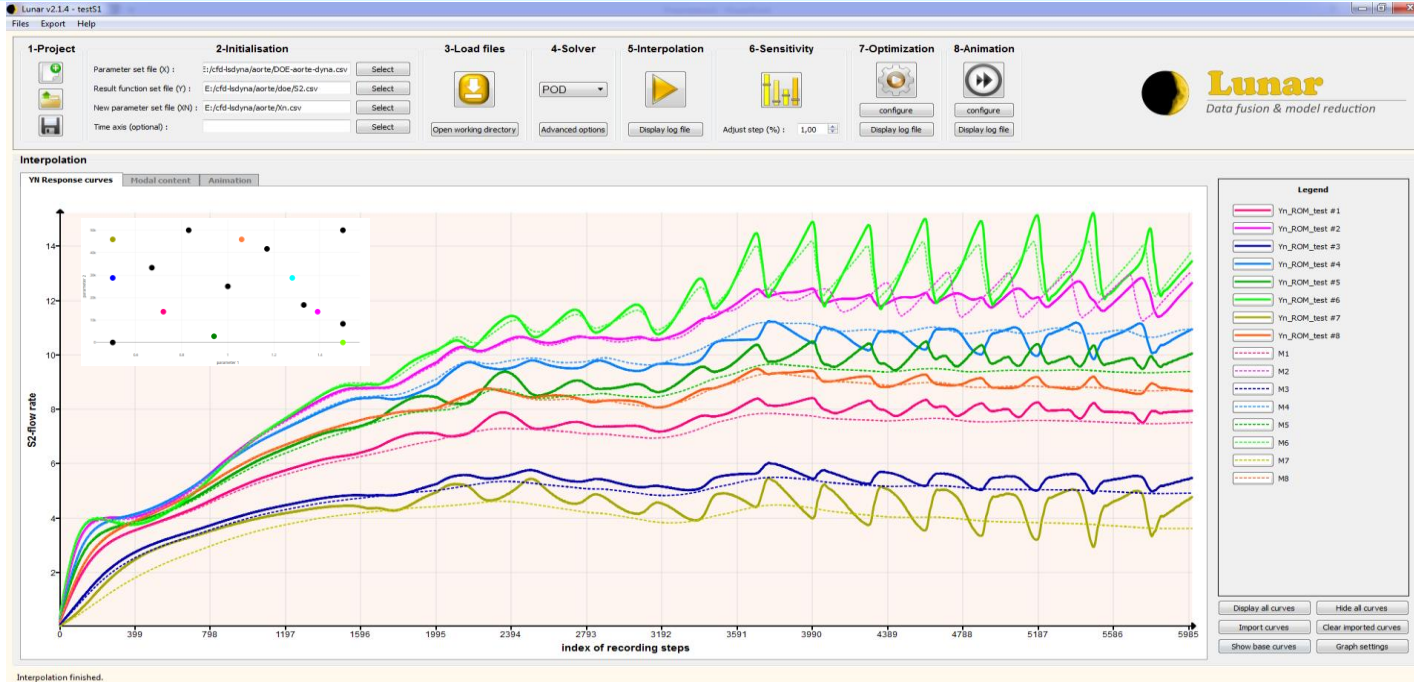
2 seconds

# Example - Fluid blood flow model



ODYSSEE

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Lunar

Data fusion & model reduction

— Lunar: 2 s

⋯ FEM : 2.5 hours



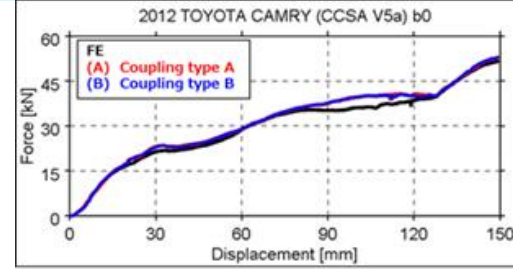
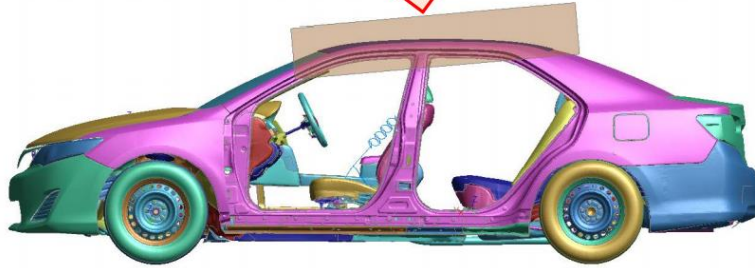
# Crash – sub-modelling

Courtesy of JSOL Corporation

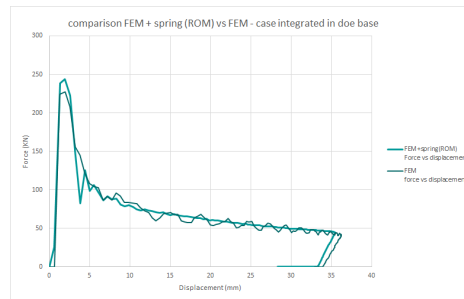
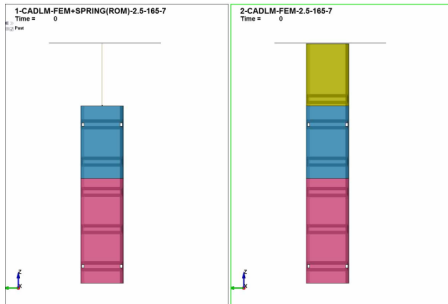
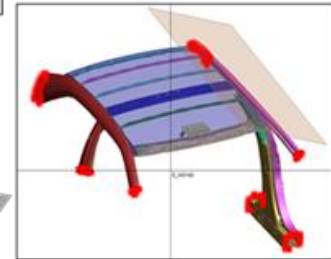
Push



Num. of node : 1,931,041  
Num. of elem : 1,920,821



	sec./cycle		
	Total	Quasar	Dyna
FY2019	13.49	13.48	0.01
FY2020	0.05	0.03	0.02
Ideal (20% of Full FE)	0.02	0.01	0.01



Sub-modelling of crash box-beam

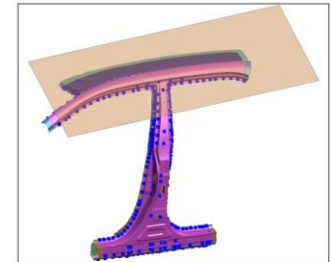
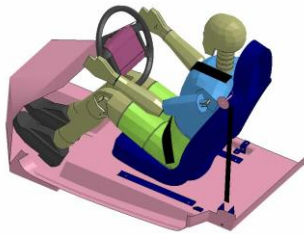


Fig1. Coupling model



# Example - Sled test

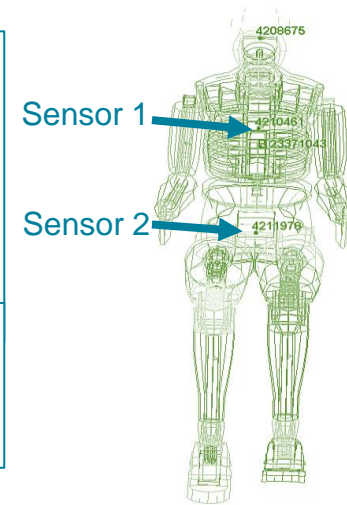
CADLM - Modele B - impact avec airbag sur mannequin  
Time 0



**15 DOE runs**

**3 Model parameters:**  
Deceleration (breaking speed)  
=> X1, X2  
Airbag mass debit  
=> X3

**Output channels:**  
Thorax compression => Y1  
Pelvis acceleration => Y2



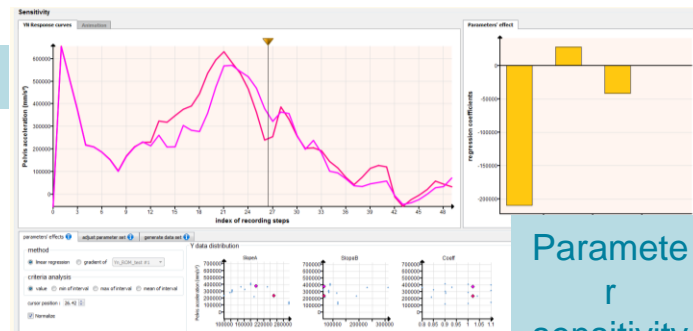
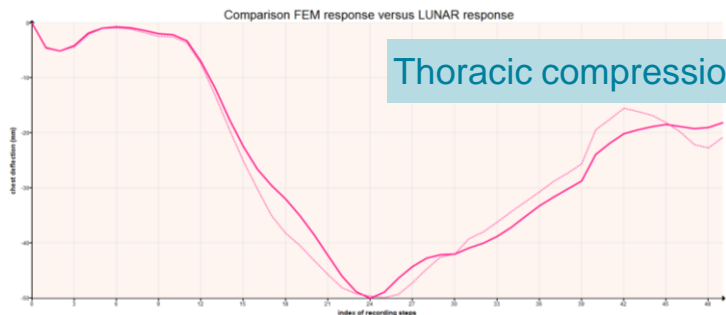
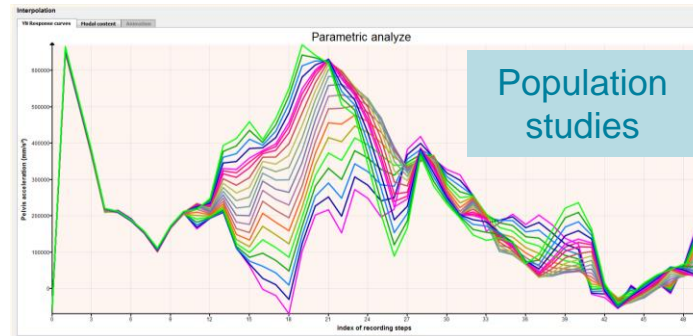
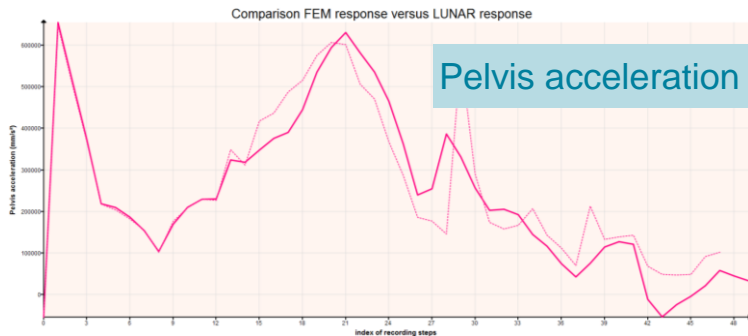
Finite Element  
solution per run

1 hour/ simulation

Lunar

1 second

# Example - Sled test



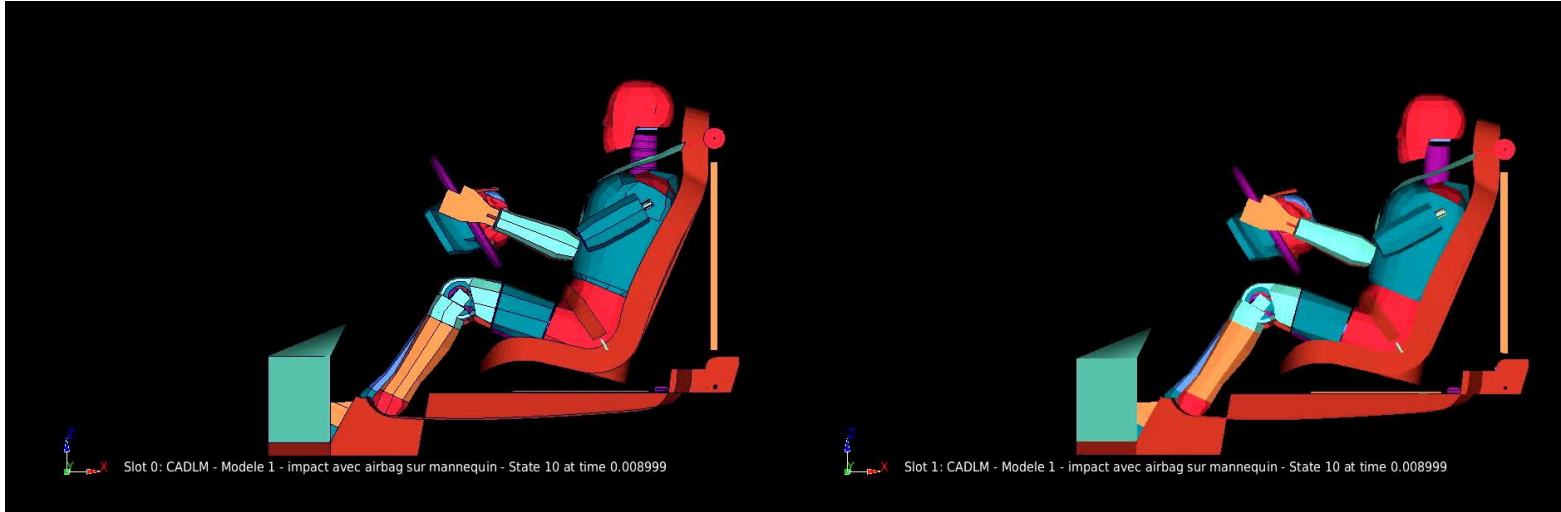
Parameter sensitivity



— Lunar: 2 s

..... FEM: 1 hour

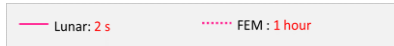
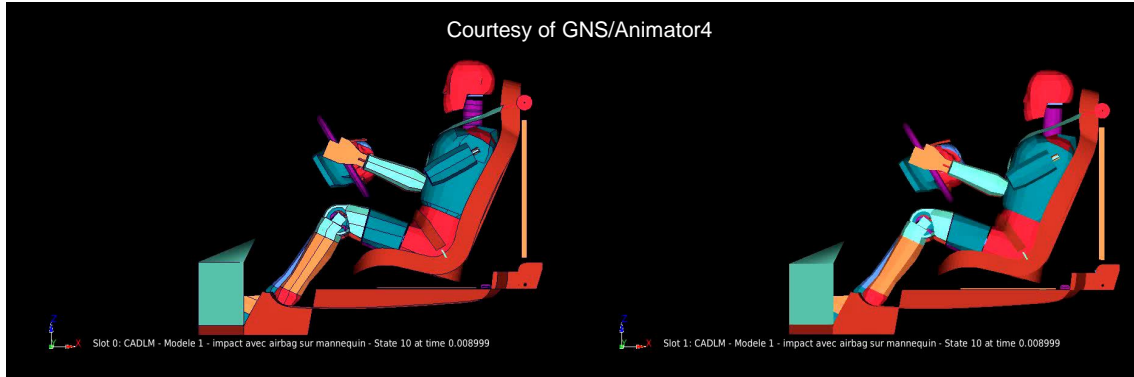
# Example - Sled test



**Animation obtained by EF (1h)**

**Animation obtained by Lunar (2s)**

# SLED TEST + AIRBAG (Optimization)



## 15 runs, 3 parameters

Deceleration (breaking speed) =>  
X1, X2

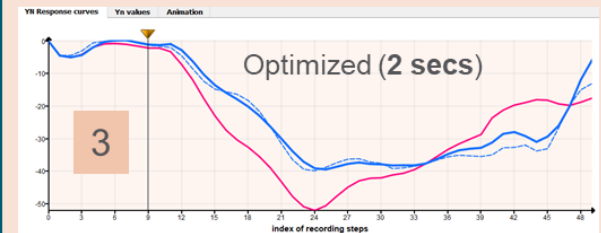
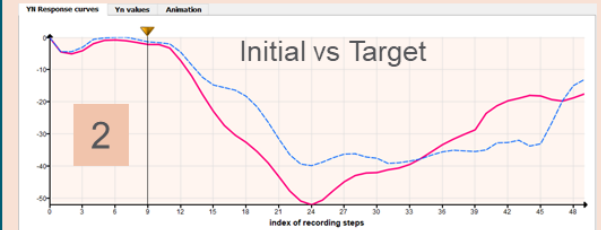
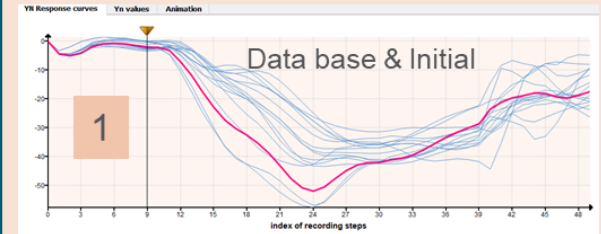
Airbag mass debit => X3

## Output Channel

Thorax compression => Y1

Pelvis acceleration => Y2

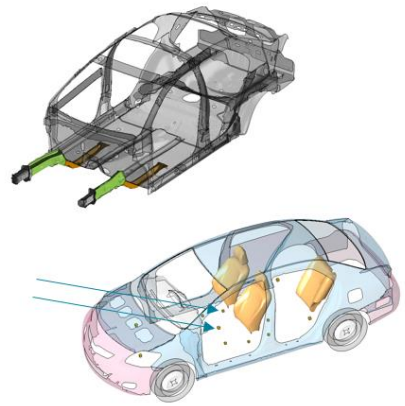
## Optimization



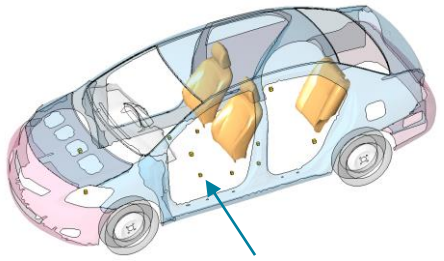
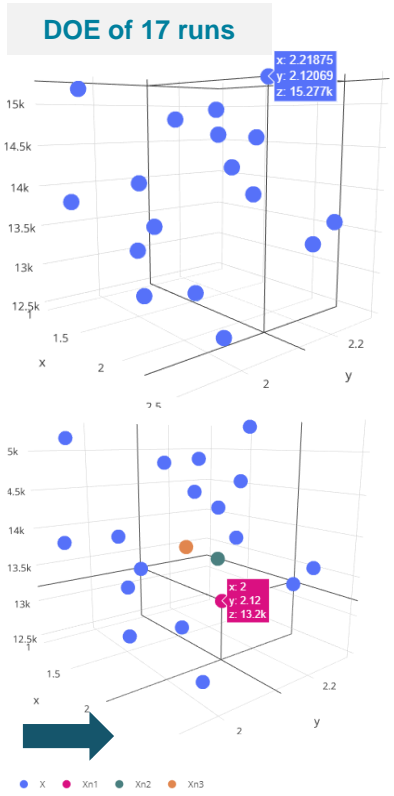
# Example – ‘Yaris Front crash’

**3 Model parameters:**  
 Rail inner thickness => X1  
 Floor support thickness => X2  
 Velocity => X3

**Output channels:**  
 Driver seat acceleration=> Y1  
 Rigid wall Force => Y2  
 Gravity center acceleration => Y3



Finite Elements	Lunar
3h per simulation	2 seconds



Driver seat accelerometer

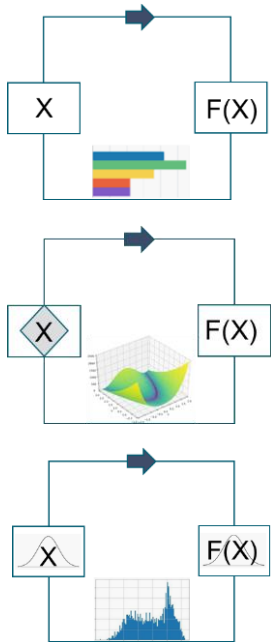
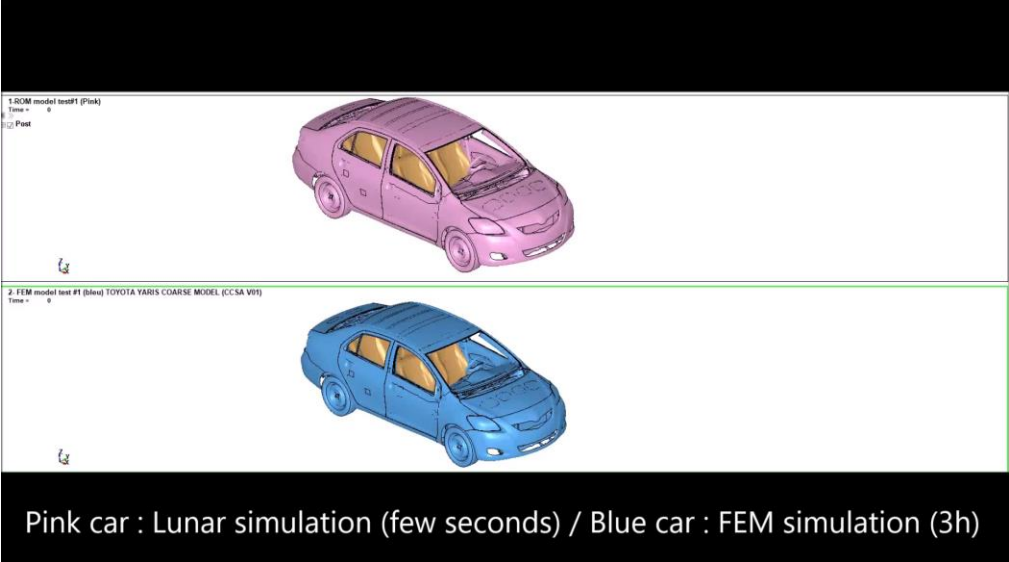
TOYOTA YARIS COARSE MODEL (CCSA V11)



393329 Nodes  
378470 Elements

	Rail inner thickness (mm)	Floor support thickness (mm)	Velocity (mm/s)
Validation 1	2,00	2,12	13200,00
Validation 2	1,60	2,20	13500,00
Validation 3	1,70	2,10	13800,00

## Driver seat acceleration



## Multi-scale & multi-physics analysis by ROM

- Require considering two phenomena with different spatial scales
- Multiphysics analysis by structure, Thermal and electromagnetic field

	Full vehicle crash		Battery short circuit	
	explicit	Explicit (structure)	Implicit (Thermal + EV)	
Solver	explicit	Explicit (structure)	Implicit (Thermal + EV)	
DT(sec.)	1.0E-6	9.0E-6	1.0E-3	
Phenomenon time (sec.)	0.4		0.1	
Num. of step	399,681	11,112	100	
COST	2h7m@128core		5min@4core	

## Model specifications



**TOYOTA YARIS**  
Nodes:1,073,273  
Elems:1,036,485

<https://www.nhtsa.gov/crash-simulation-vehicle-models>



**BATTERY CELL**  
Nodes:4,331  
Elems:3,618

## Variables

- IN : Module plate thickness
- OUT : Battery cell deformation

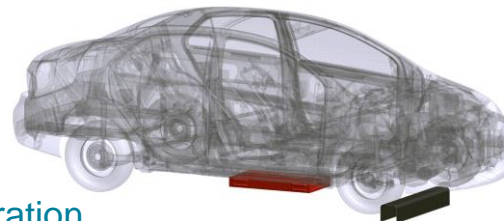
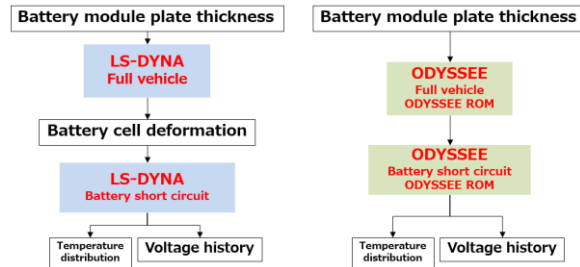
## Datasets

- Thickness 0.4 - 2.0 mm, 5 sampling

## Costs

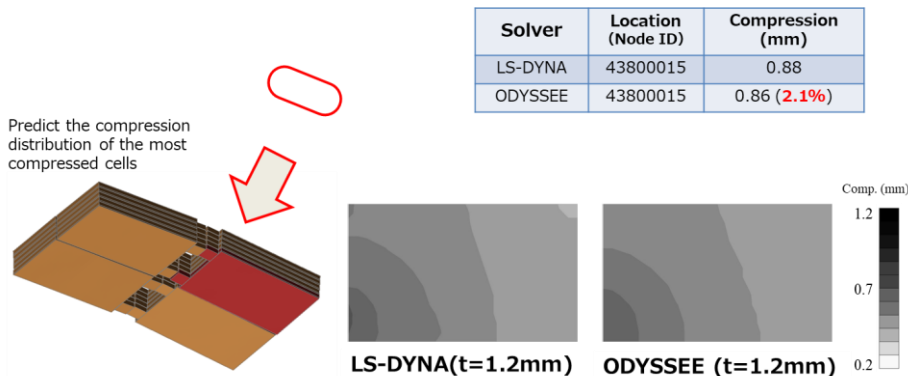
- Learning : 0.3 sec
- Predict : 0.4 sec

## Flow of coupling analysis



Courtesy of JSOL Corporation

# Battery short circuit analysis (Image)



## Variables

- IN : Battery cell deformation (**Picture**)
- OUT : Temperature distribution

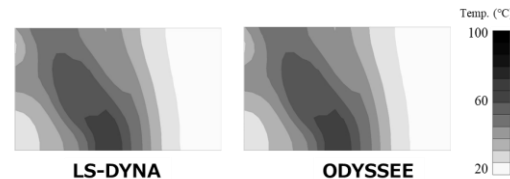
## Datasets

- 15 sampling

## Costs

- Learning : 96 sec
- Predict : 0.3 sec

Solver	Location (Node ID)	Max. Temperature (°C)
LS-DYNA	43800104	89.3
ODYSSEE	43800104	89.1 ( <b>0.2%</b> )



## Variables

- IN : Battery cell deformation (**Picture**)
- OUT : Terminal voltage

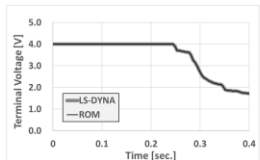
Solver	Terminal voltage (V)
LS-DYNA	1.71
ODYSSEE	1.72 ( <b>0.1%</b> )

## Datasets

- 15 sampling

## Costs

- Learning : 96 sec
- Predict : 0.3 sec



- It is possible to dramatically reduce the simulation COST by using ODYSSEE ROM.
- Coupling between different solvers becomes easy by focusing on input / output only.

Solver	Full vehicle Analysis (Structure)		Battery short circuit (Structure+Thermal+EM)	
	COST (sec/core)	Error (%)	COST (sec/core)	Error (%)
ODYSSEE (Learning)	0.3	2.1	96	0.2
ODYSSEE (Predict)	<b>0.4</b>		<b>0.3</b>	
FEM	975,360	-	1200	-

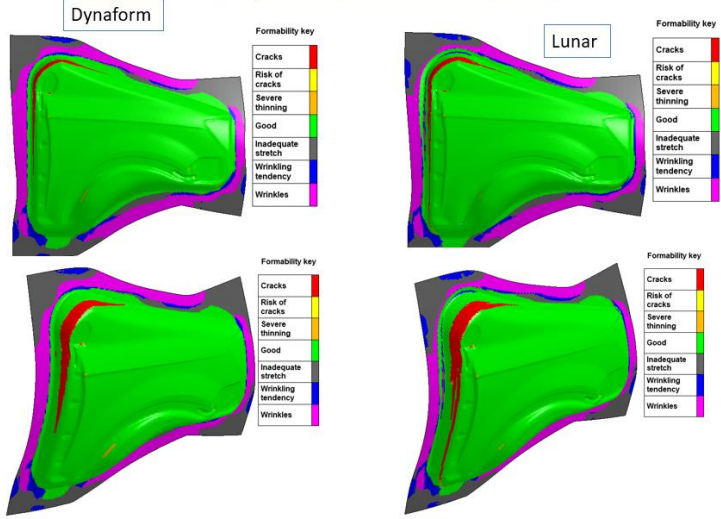
Courtesy of JSOL Corporation



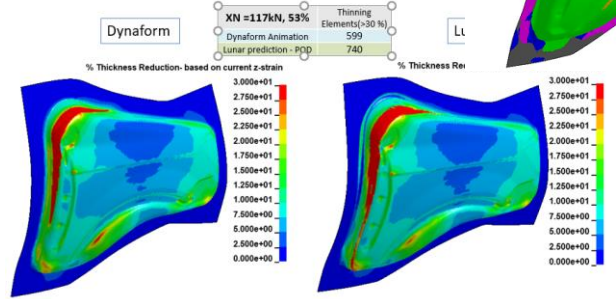
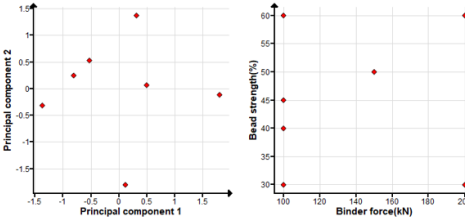
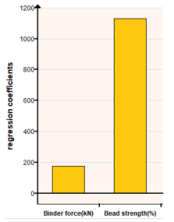
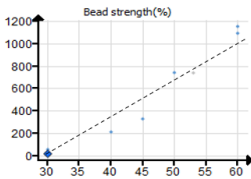
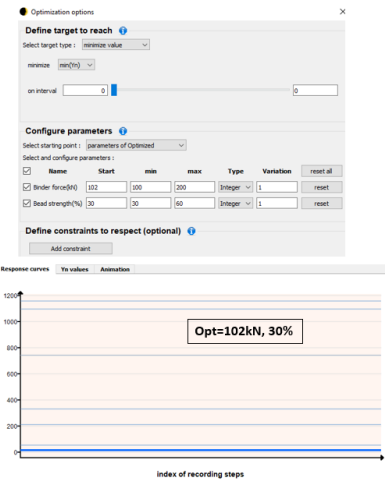
	Input (X)		Output (Y)		
	Binder Force (kN)	Bead Strength (%)	Crack Elements %	Crack area%	No Of elements
M1	100	30	0.0342	0.051	22
M2	200	30	0.0996	0.169	64
M3	100	60	0.948	4.45	609
M4	200	60	0.898	5.08	577
M5	100	40	0.263	0.625	169
M6	100	45	0.336	1.157	216
M7	150	50	0.455	2.195	292
M8	117	53	0.486	2.494	312

### Animation Comparison

XN = 117kN, 53%	Crack Elements %	Crack area%	No Of elements
Dynaform Animation	0.486	2.494	312
Lunar prediction - POD	0.554	2.61	356



### Formability Optimization



Bender force, Bender strength  
7 runs only

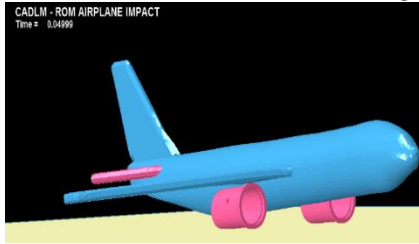
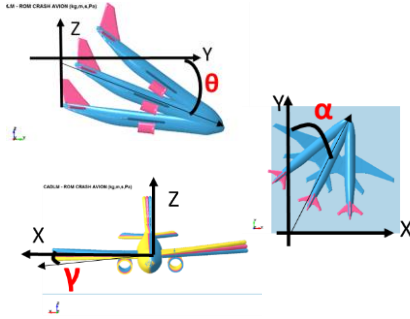
**Initial:**  
XN = 117kN,  
53%



**Optimal:**  
Xopt=102kN,  
30%

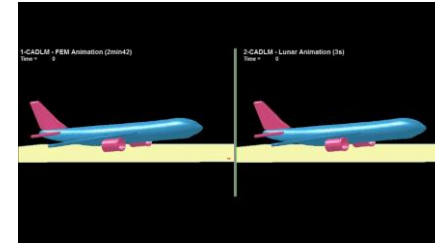
Courtesy of ETA

# Flight Simulator (landing conditions)



<https://www.youtube.com/watch?v=oNQ0wT0WFn>

g



15 runs, 3 parameters

- Angle  $\theta$   $\Rightarrow$  X1
- Angle  $\alpha$   $\Rightarrow$  X2
- Angle  $\gamma$   $\Rightarrow$  X3

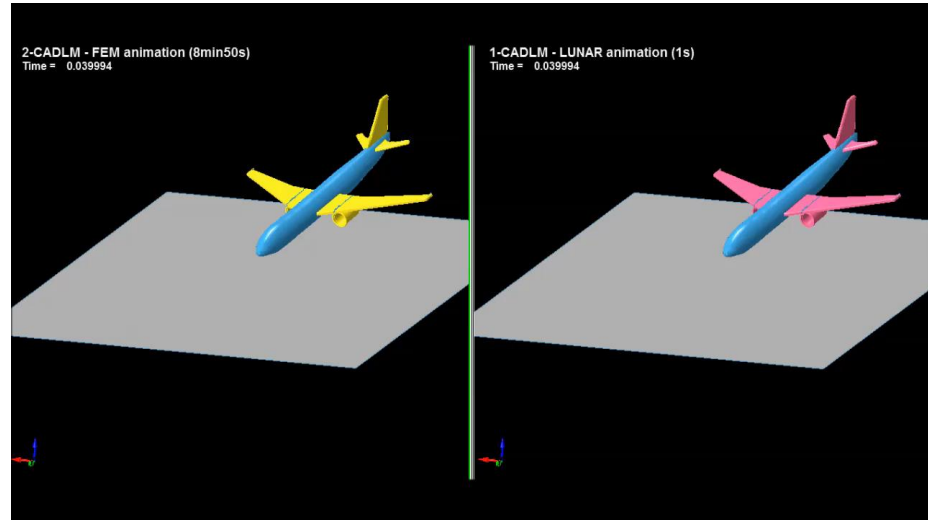
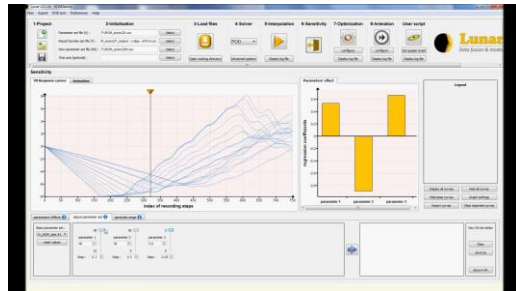
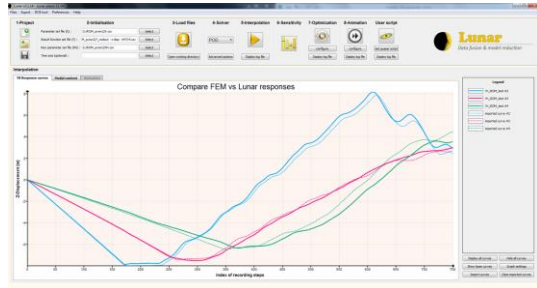
Output Channel

- Nose z-displacement  $\Rightarrow$  Y1

Elapsed Time

- FE  $\Rightarrow$  9 minutes
- LUNAR  $\Rightarrow$  1 sec

# Flight Simulator (landing conditions)

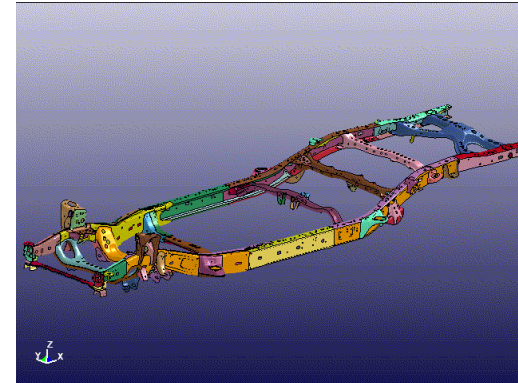
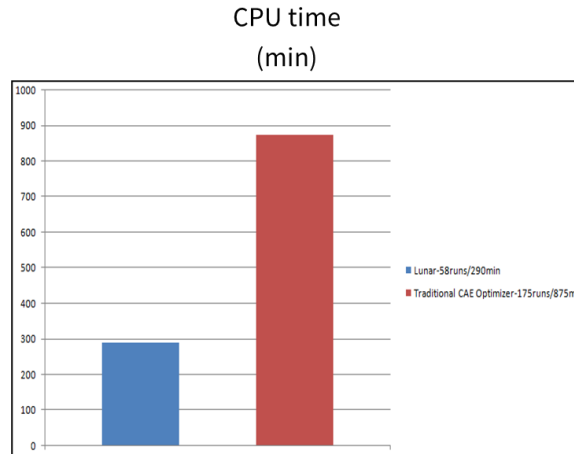


# Chassis Frame Optimization for Natural Frequency ODYSSEE

Explore new industrial horizons

- Lunar enabled to reduce the DOE size by 67% (58 runs against 175 runs) and still predict response with good accuracy.
- The CPU time in this case is 33% of the traditional optimization approach.
- For a problem involving more design variables and multiple loadcases, Lunar offers more substantial time and CPU cost savings.
- Once the predictions are validated, Lunar's ROM approach can replace the CAE method to calculate response for any design combination and save significant process time.

Design	First Natural Frequency	Total mass	Rear Stub Mass	Mass Saving (%)
Baseline	11Hz	245	112	
Lunar Optimization	10Hz	224.4	91.4	18.4%
CAE Validation	10.3Hz	223.3	90.3	19.3%



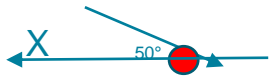
Courtesy of ETA

# Fusion for image processing – Automotive head Impact analysis

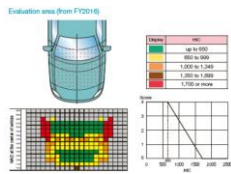
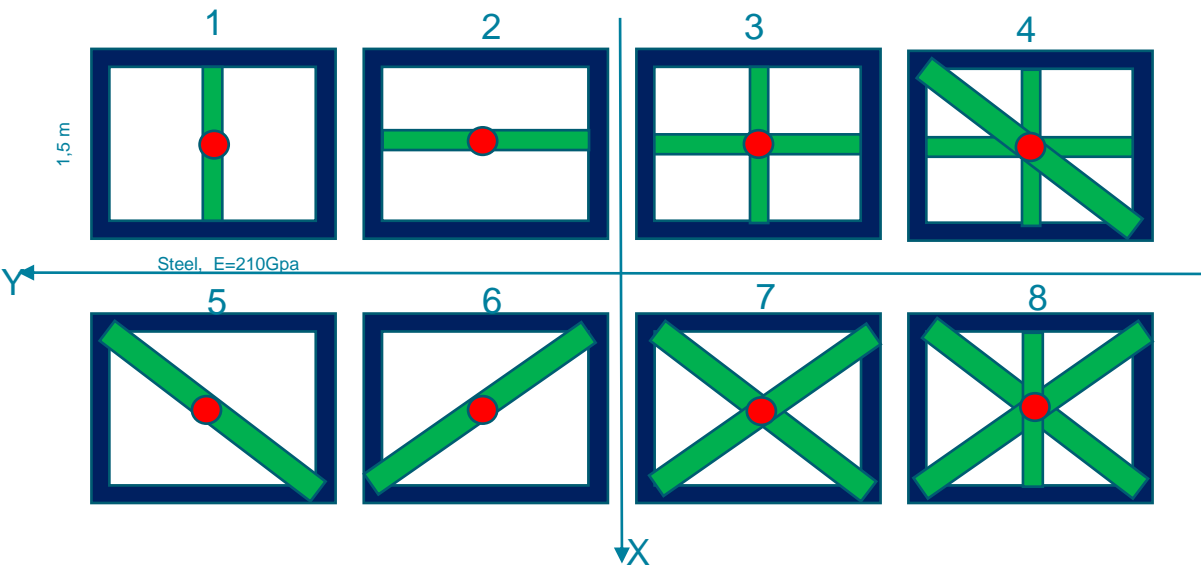
## Child head Impact on bonnet

Frame- Steel  $E=210$  Gpa; Blue part 3mm; green part 10 mm

Fixed on 4 corners ; Impactor mass=3.5kg,  $V_x=40$  km/h



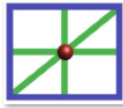
Child head (sphere; Diameter ~25 cm ?)



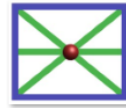


Explore new industrial horizons

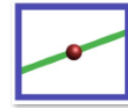
# ODYSSEE Reduced Order Modeling for image processing - Accelerations



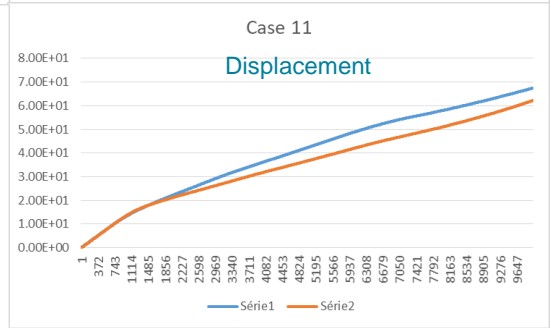
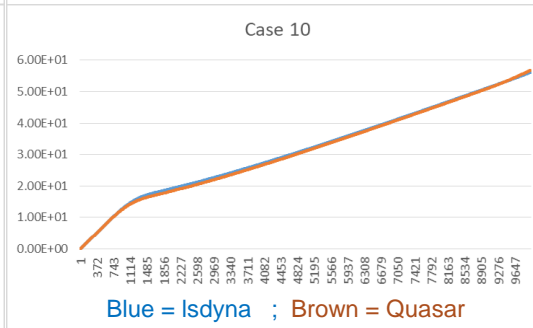
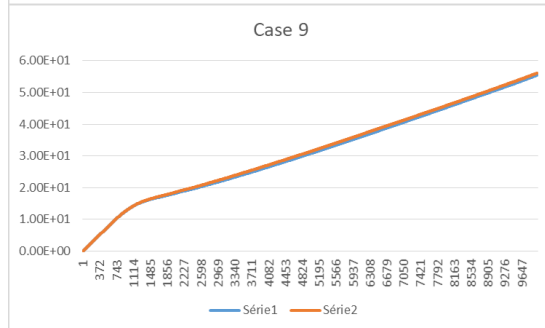
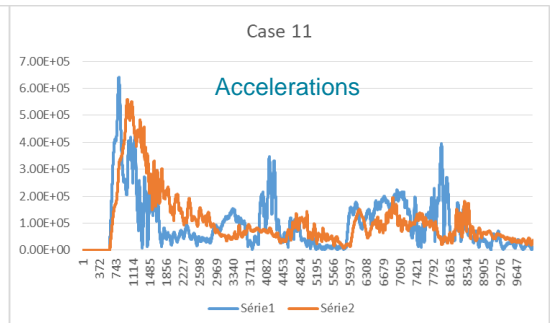
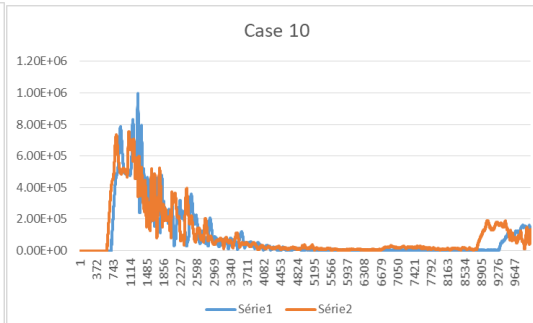
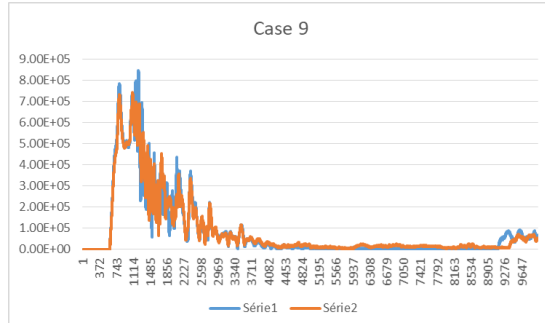
Case9.jpg



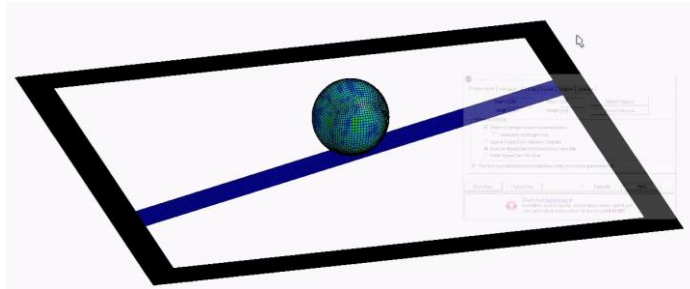
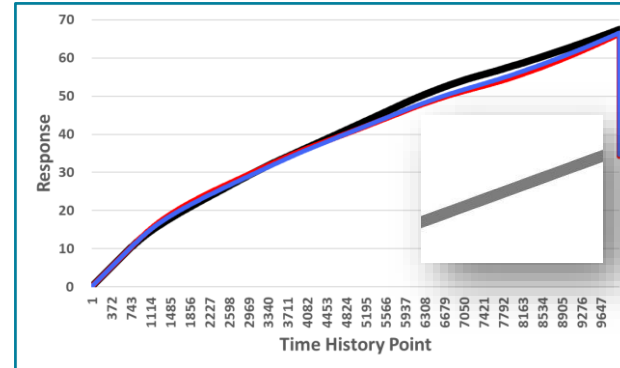
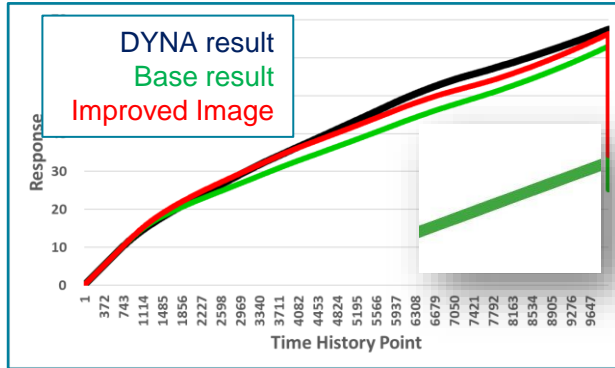
Case10.jpg






Case11.jpg



## Influence of image boundaries

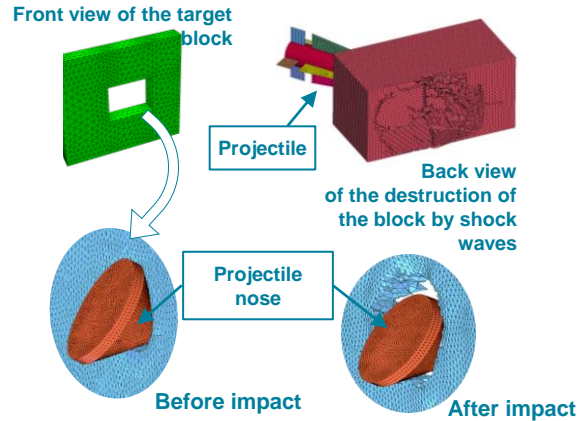


Courtesy of Masahiro Takeda, JSOL

	<b>Error [%]</b>	<b>Max Relative Error [%]</b>
	5.63	9.69
	2.08	4.5
	1.70	3.67

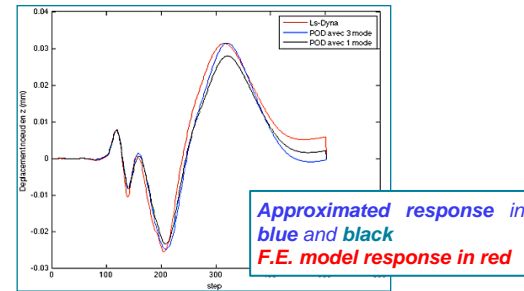


Costly to run a complex dynamic and non linear model like ballistic impact



**Methodology** for ALE based reduced model  
**Real time solution** for simulators based on FE  
**Reduced CPU cost** by reducing the need for finite element solutions

- **Reduced model** of ALE model (fluid/structure interaction)
- Application for real-time predictive **training simulator**
- **Combine reduced model** with a finite model using domain decomposition techniques







# ODYSS

Explore new industrial horizons

# EF

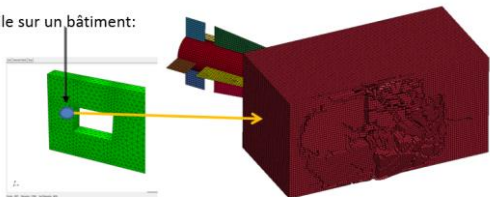
# Ballistic impact with ALE simulations

(Defense sector)

Collaboration SILKAN/CADLM

## Le Scénario

➤ Impact missile sur un bâtiment:

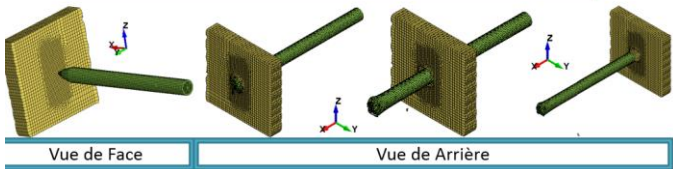


CCP n°2 - 16/03/2015

### Missile Fleche

#### Modèle EF Réduit Centre

Evolution de l'impact au centre de la cible

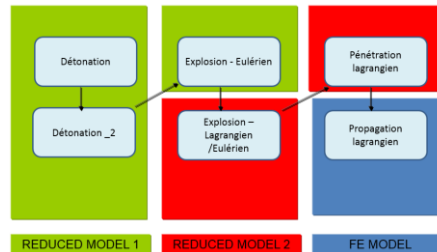


Vue de Face

Vue de Arrière

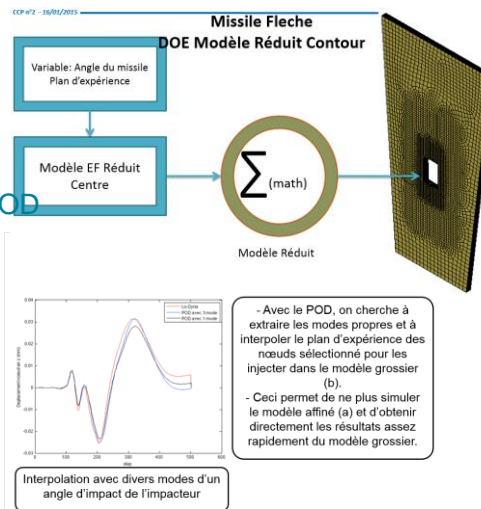
Complexe  
➔  
Longue

## Workflow de la simulation



➔  
EF vers base POD

➔  
Workflow





Reduce BIRD strike analysis simulation time using **LUNAR** (ROM) and **NOVA** (Optimization)

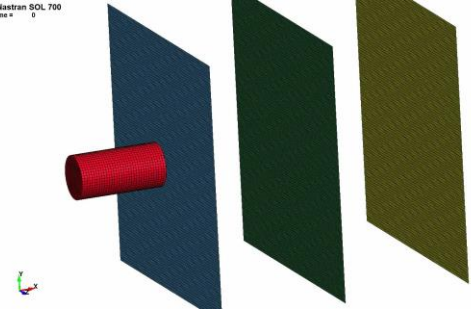
Target:

- Find the optimal thickness distribution for plate1, plate2 and plate3

Constraint:

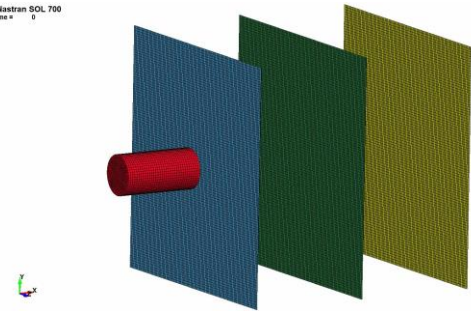
- No elements erosion in plate3

Nastran SOL 700  
Time = 0



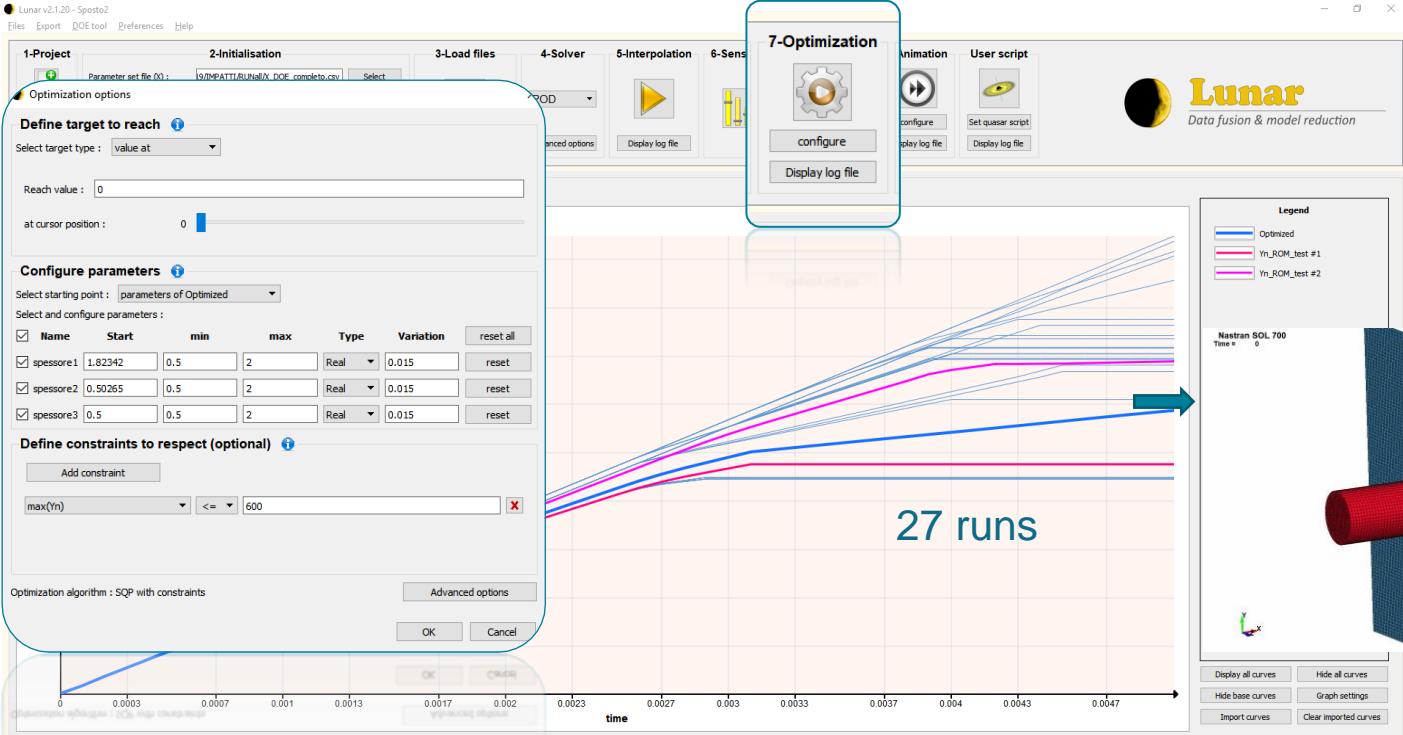
*Minimum Thickness 0.5 mm*

Nastran SOL 700  
Time = 0



*Maximum Thickness 2.0 mm*

# Lunar : Optimization



The screenshot displays the Lunar software interface with several key components:

- 7-Optimization Panel:** A floating window with a gear icon and buttons for 'configure' and 'Display log file'.
- Optimization options dialog:** A large dialog box on the left with the following sections:
  - Define target to reach:** 'value at' selected, 'Reach value: 0', 'at cursor position: 0'.
  - Configure parameters:** 'parameters of Optimized' selected. A table of parameters:

Name	Start	min	max	Type	Variation	reset all
spessore1	1.82342	0.5	2	Real	0.015	reset
spessore2	0.50265	0.5	2	Real	0.015	reset
spessore3	0.5	0.5	2	Real	0.015	reset
  - Define constraints to respect (optional):** 'Add constraint' button, 'max(Yn) <= 600'.
  - Algorithm:** 'SQP with constraints'.
- Graph:** A line graph showing '27 runs' over 'time'. The x-axis ranges from 0 to 0.0047. Multiple lines represent different runs, with a blue arrow pointing to the right.
- 3D Model:** A mechanical assembly with a red cylinder and three plates (blue, green, yellow).
- Legend:** 'Optimized' (blue line), 'Yn\_ROM\_test #1' (pink line), 'Yn\_ROM\_test #2' (magenta line).
- Mastran SOL 700:** 'Time = 0'.
- Buttons:** 'Display all curves', 'Hide all curves', 'Hide base curves', 'Graph settings', 'Import curves', 'Clear imported curves'.

ELAPSED time = 15.42 seconds

# Chassis Frame Optimization for Natural Frequency ODYSSEE

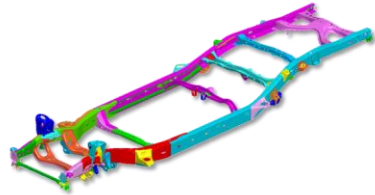
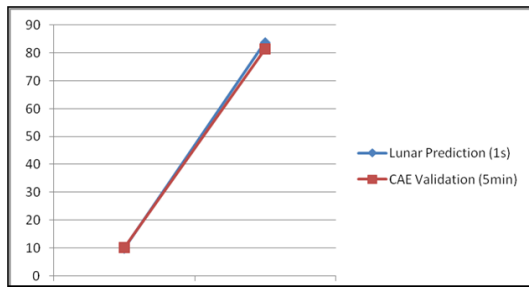
Explore new industrial horizons

**Design details and constraints**

- Mass is 245kg
- First Torsional mode is at 11Hz
- Only the rear stub (mass 112kg) needs to be optimized so that the crash performance is not altered.

**Optimization Targets**

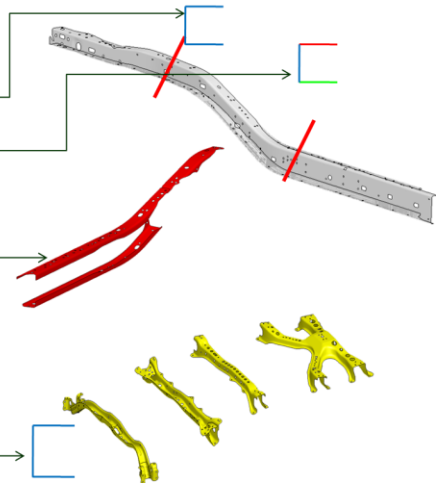
- First Torsional mode should be within 10% limit of Baseline design ( $\geq 10\text{Hz}$ )



**Rear Stub (to be optimized):**  
 Mass = 112 kg  
 (of Rail, X Members and reinforcements)  
**Full Frame Mass**  
 245 kg

Frame Rear Stub components were parameterized for Geometry, Grade and Gauge.

Variables
Rail divided into three sections to vary the section individually (Front, middle, rear)
TWB Rail (three) with separate thickness of flanges (Each TWB will have 3 section thickness variable)
TWB Rail (grade)
Reinforcements (gage/grade)
Sections of X-members: Width and height of each member individually (no transition 15%)
4 cross members (gage/grade)
X-member Brackets gage/grade



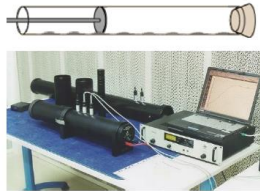
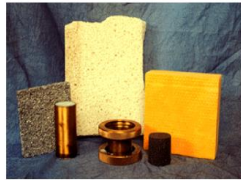
	Frequency	Mass
Lunar Interpolation	9.9Hz	83.8Hz
CAE Validation	10.3Hz	81.5Hz

Courtesy of ETA



# ACTRAN - Predicting the absorption coefficient of a porous material

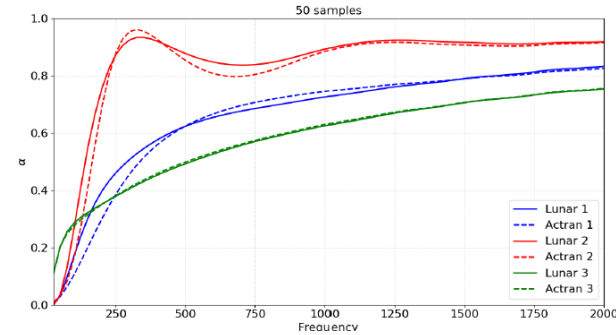
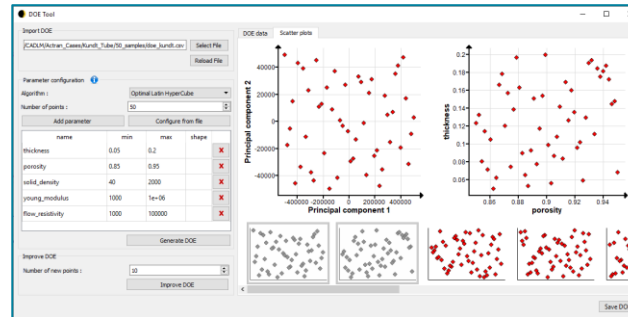
$$\alpha = 1 - \left| \frac{P_{reflected}(\omega)}{P_{incident}(\omega)} \right|$$



Experimental Setup

Quantity	Min	Max
Thickness [m]	0.05	0.2
Porosity [m]	0.85	0.95
Solid density [kg/m^3]	40	2000
Young's modulus [Pa]	1000	1000000
Flow resistivity [Ns/m^4]	1000	100000

Numerical Setup

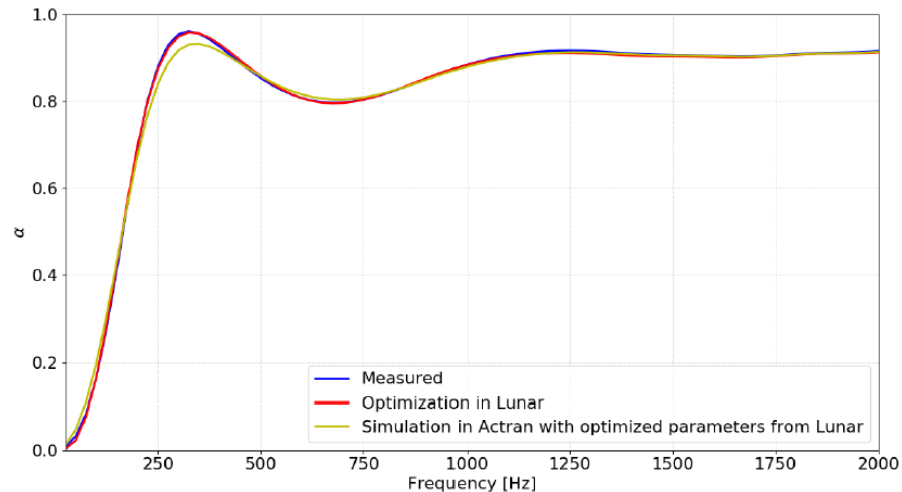


Validation

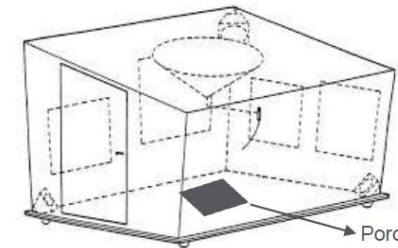
Courtesy of MSC

## Optimization for finding material properties

- A dummy measured curve is given and the material properties are found via optimization
- Then the optimized parameters are used to create a new Actran model in order to compare the optimized model with reality
- The DOE with the 50 samples is used as a starting point (time taken: 3 min)



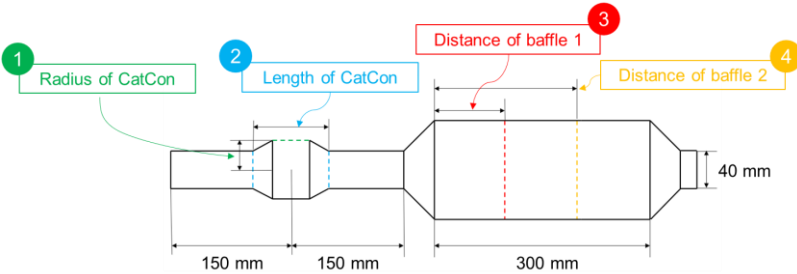
*Multi-layer trim component*



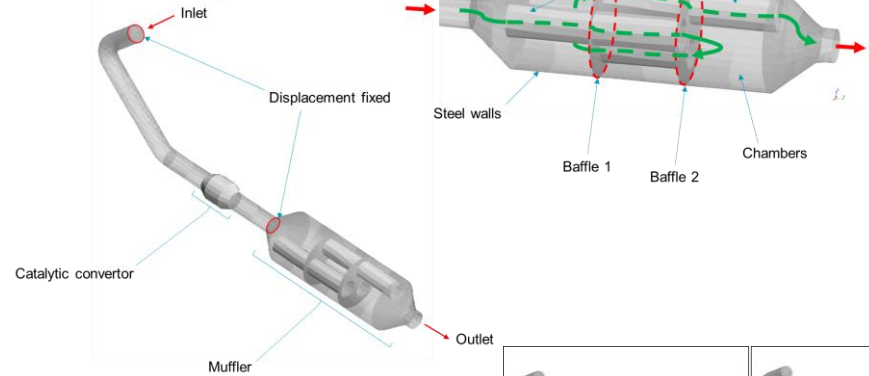
*Alpha cabin*

Porous sample

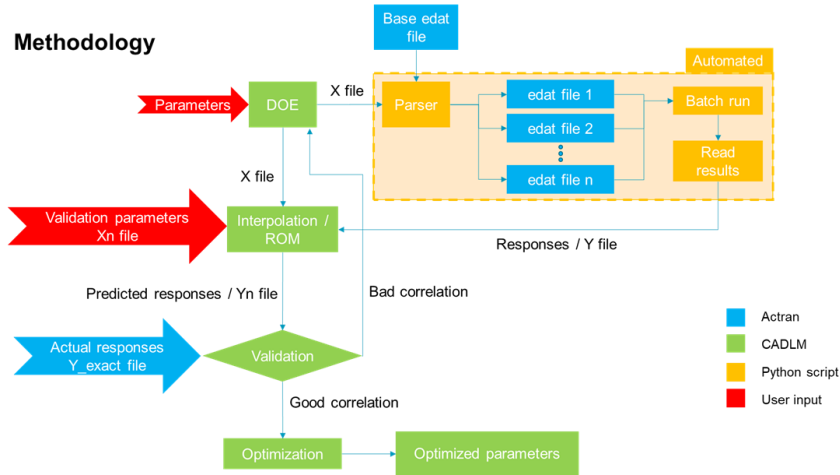
# Actran : Exhaust System Optimization



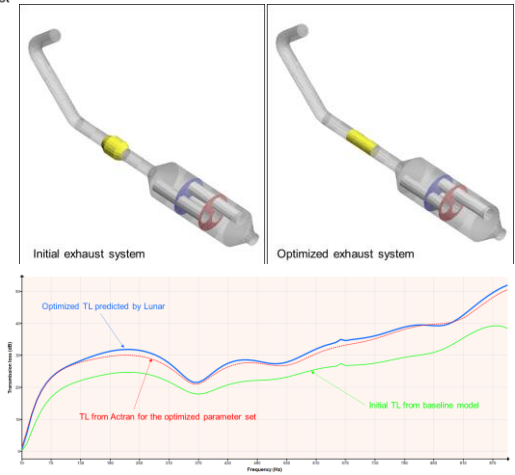
## Base model setup in Actran



## Methodology

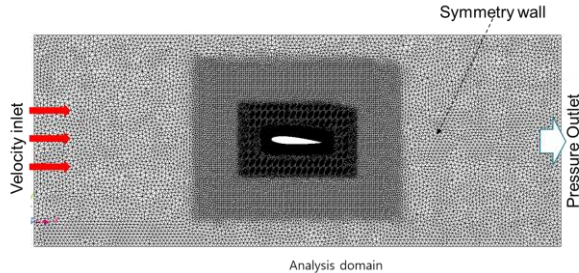


## Optimized solution



## AOA MODEL AND ANALYSIS CONDITIONS

- Tetra Grid settings



- Domain size
  - 1.5 x 0.6 x 0.015 [m]

- Mesh 정보

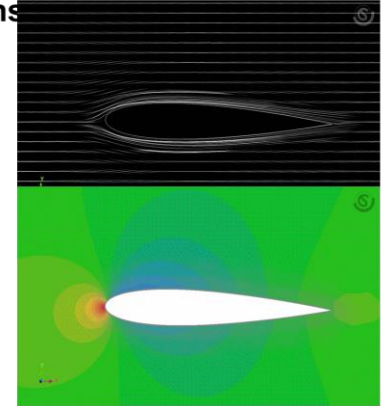
	해석 도메인
Mesh size	0.32 ~ 10 [mm]
Mesh number	3,789,135 개

> Reference : "Flying hot wire study of flow past an NACA 4412 airfoil at maximum lift, D. Coles and A.J. Wadcock, AIAA J, 17, 321-328 (1979)"

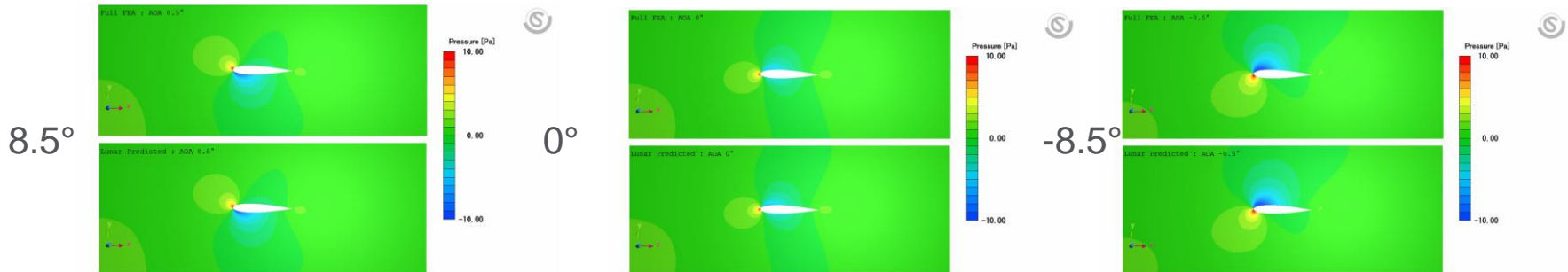
## Set AOA model and analysis conditions

- AOA (Angle of Attack) analysis model
- Analysis conditions

Setting analysis conditions	
Turbulence Model	SST kw Model
Steady/Transient	Steady analysis
Density	1.206 kg/m <sup>3</sup>
Viscosity	1.83e-05 Pa s
Pressure Correction Method	SIMPLEC
Time Derivative Terms	Second order implicit scheme
Airfoil Model	NACA 4412 airfoil



Flow around NACA 4412 airfoil

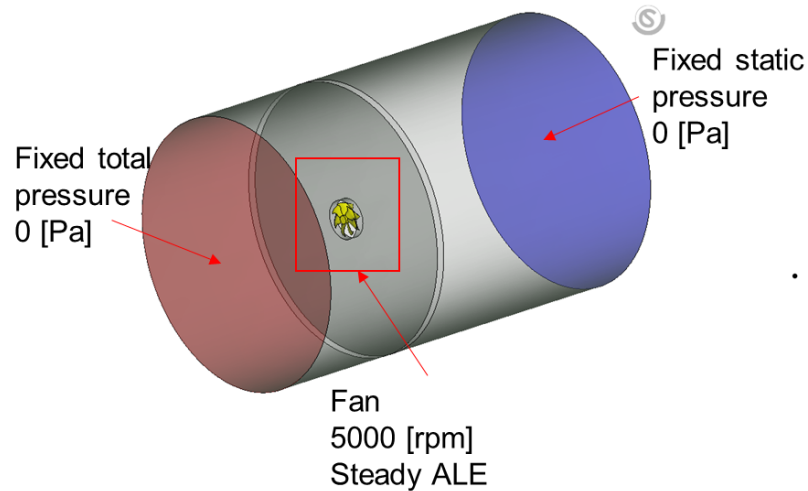


Predictions vs Full CFD



## scFLOW Steady-State analysis

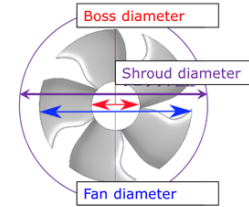
- Prediction of flow rate by fan shape



- Prediction of flow rate by fan shape
  - Design Variables and Objective Function

Design Variable	Unit	Min	Max
Number of wings	Sheet	3	10
Shroud diameter	Mm	100	130
Fan diameter	mm	100	120
Boss diameter	mm	30	40

Objective function	Unit
Outlet mass flow rate	kg/s

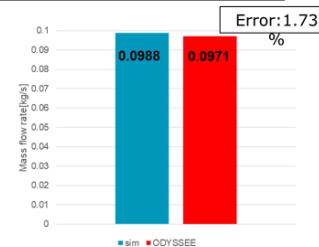


- Sampling : Optimal Latin Hyper Cube n=30
- Comparing scFLOW and Lunar in some design variable combinations.

### Result

- Comparing scFLOW and Lunar in some design variable combinations.

Difference between the analyzed value and the predicted value

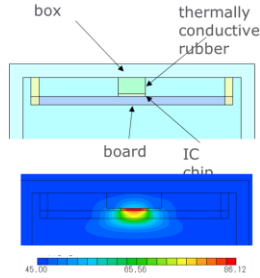
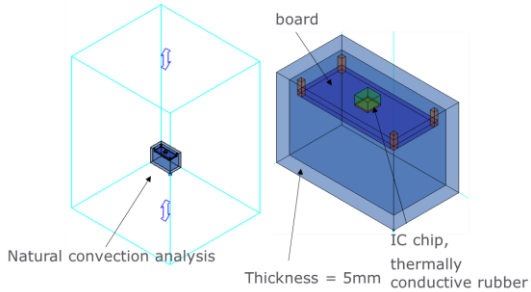


Calculation time  
scFLOW : ~16min (72Core)  
ODYSSEE : Few seconds

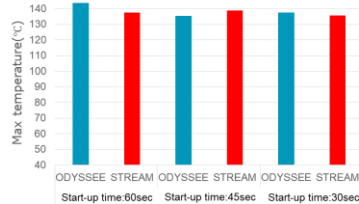
Courtesy of Tomoyuki Hirabayashi (MSC Japan)

## scSTREAM Transient analysis

- Prediction under unsteady heating conditions
  - Predicts the maximum temperature of the IC chip



- Result
  - Max temperature and calculation time



Match in the range of  $\pm 0.6^\circ$

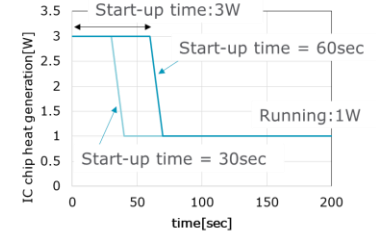
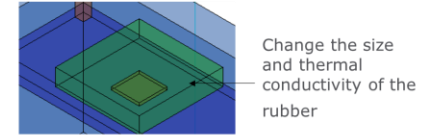
Calculation time  
scSTREAM :58 sec  
(72Core)  
ODYSSEE : Few seconds

## scSTREAM Transient analysis

- Prediction under unsteady heating conditions

Design Variable	Unit	Min	Max
Rubber size	mm	10	30
Rubber thermal conductivity	W/(m · K)	0.8	7.3
IC Chip High heat generation Start-up time	sec	30	60

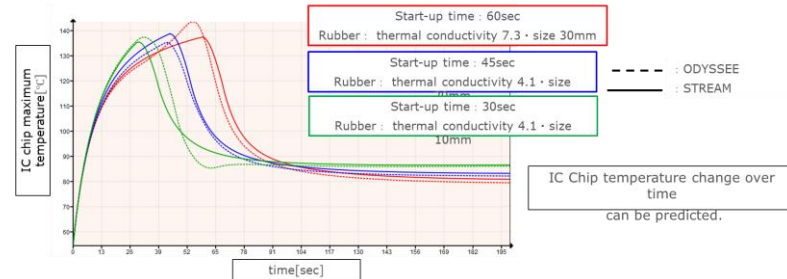
- Objective function: IC chip maximum temperature[°C]
- Sampling : Optimal Latin Hyper Cube n=20



## scSTREAM Transient analysis

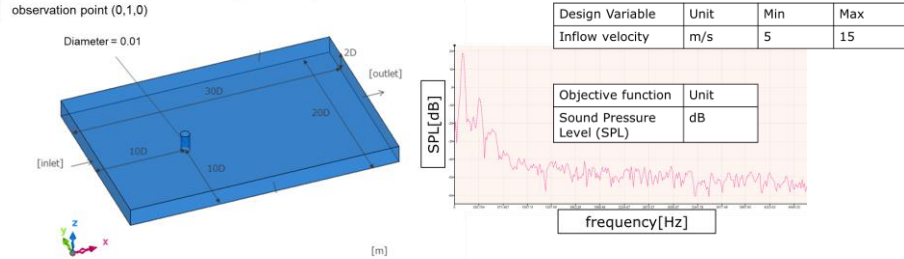
Reduce heat generation by shortening the start-up time.

- Result
  - Predicting the maximum temperature of an IC chip



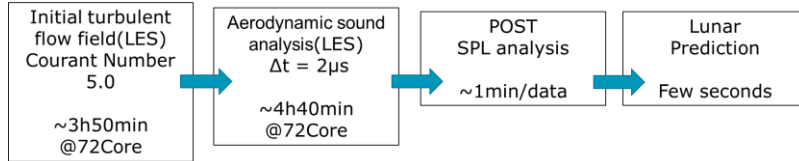
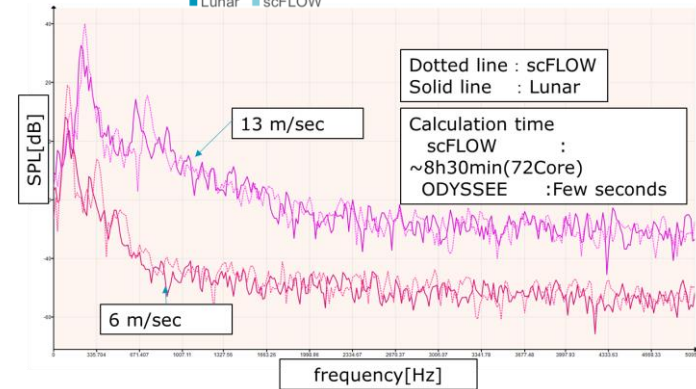
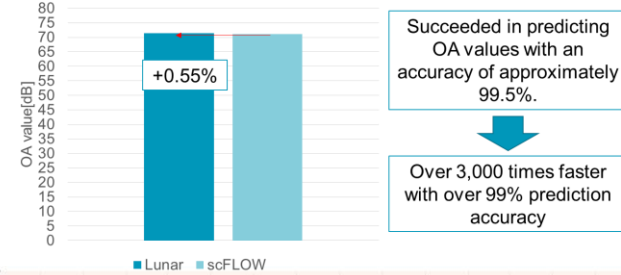
## scFLOW Prediction of aerodynamic sound analysis

- Evaluation of aerodynamic sound around a three-dimensional cylinder
  - It predicts the aerodynamic sound when the inflow velocity is changed.



### Result : OA value

- Comparison of scFLOW and ODYSSEE , with training in OA value(overall value, Total sound pressure)
- 13m/sec conditions were evaluated.
- The predicted deviation of OA values was less than 1%, which was highly accurate.

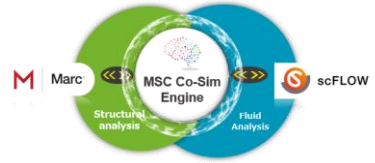
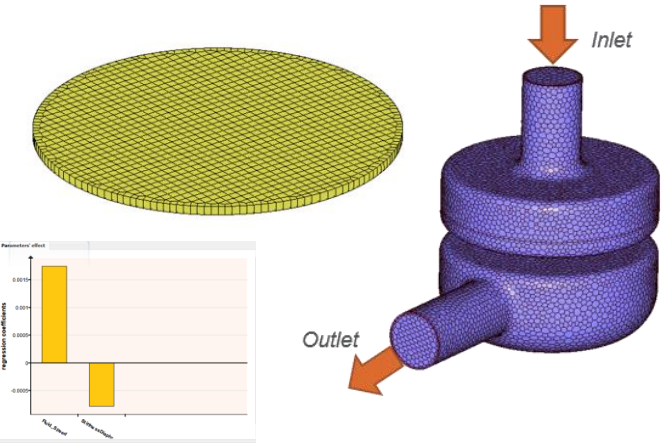


Courtesy of Tomoyuki Hirabayashi (MSC Japan)

## Design of experiments

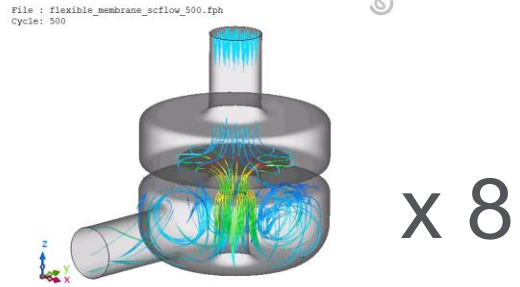
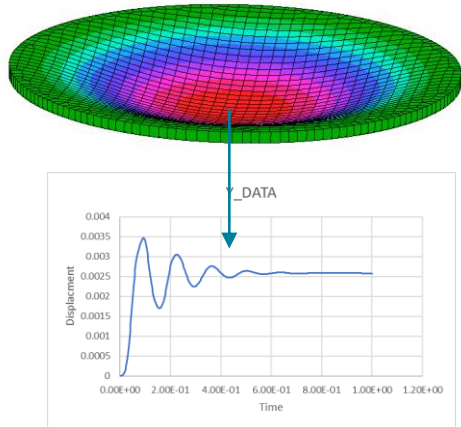
**X: (2 variables)**

- 1. Inlet Fluid Speed
- 2. Membrane Stiffness

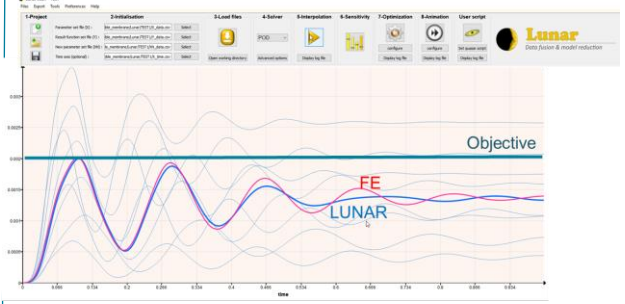


**Y:**

- 1. Displacement center of membrane

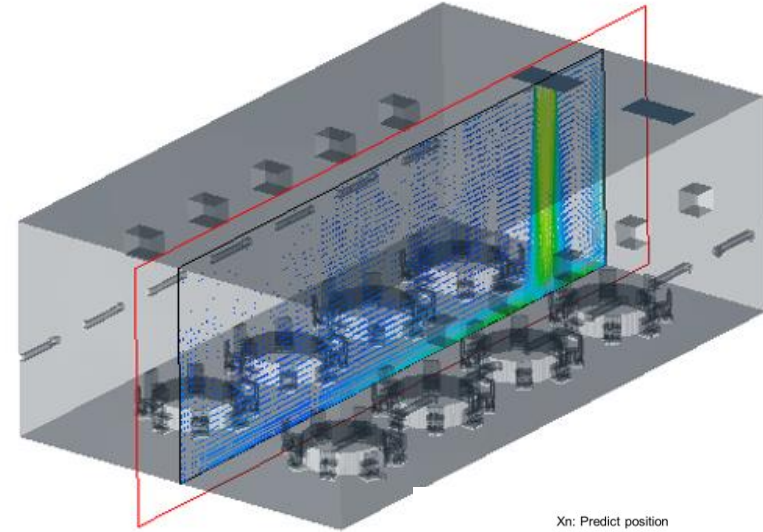


## Result Verification

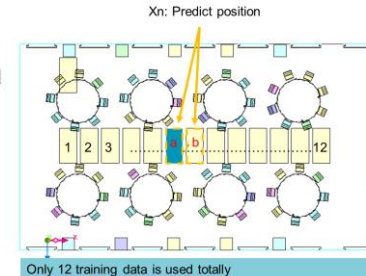


# Restaurant Airconditioning

- Inflow position is constant
- transient analysis: 0~200s
- Design variable: inflow velocity
- Prediction: 1.25, 1.3(=M3), 1.45 m/s
- # sampling: 6 case (1, 1.1, 1.2, ... 1.5m/s)



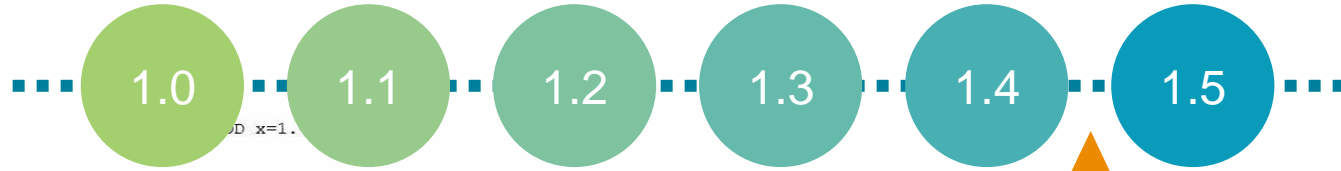
cross section



xn1 xn2

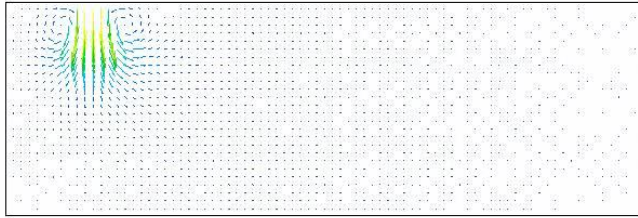


# Restaurant Airconditioning – Optimal positioning

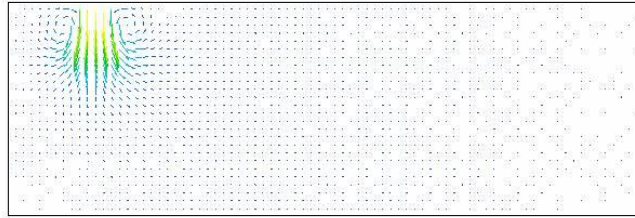


CFD on xn3 param  
Time : 5.000000

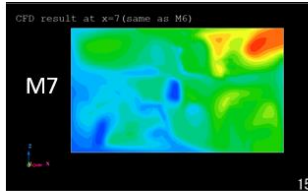
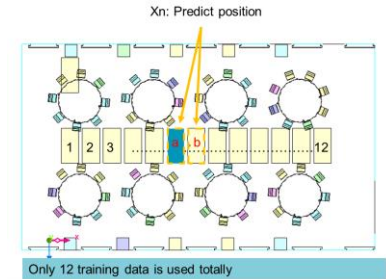
CFD



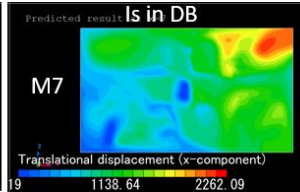
LUNAR



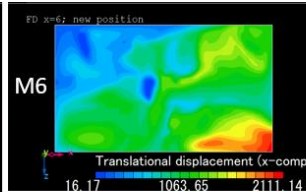
•  $x=1.45$



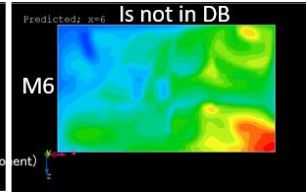
CFD



Neighbour = 1, pow = 1.0

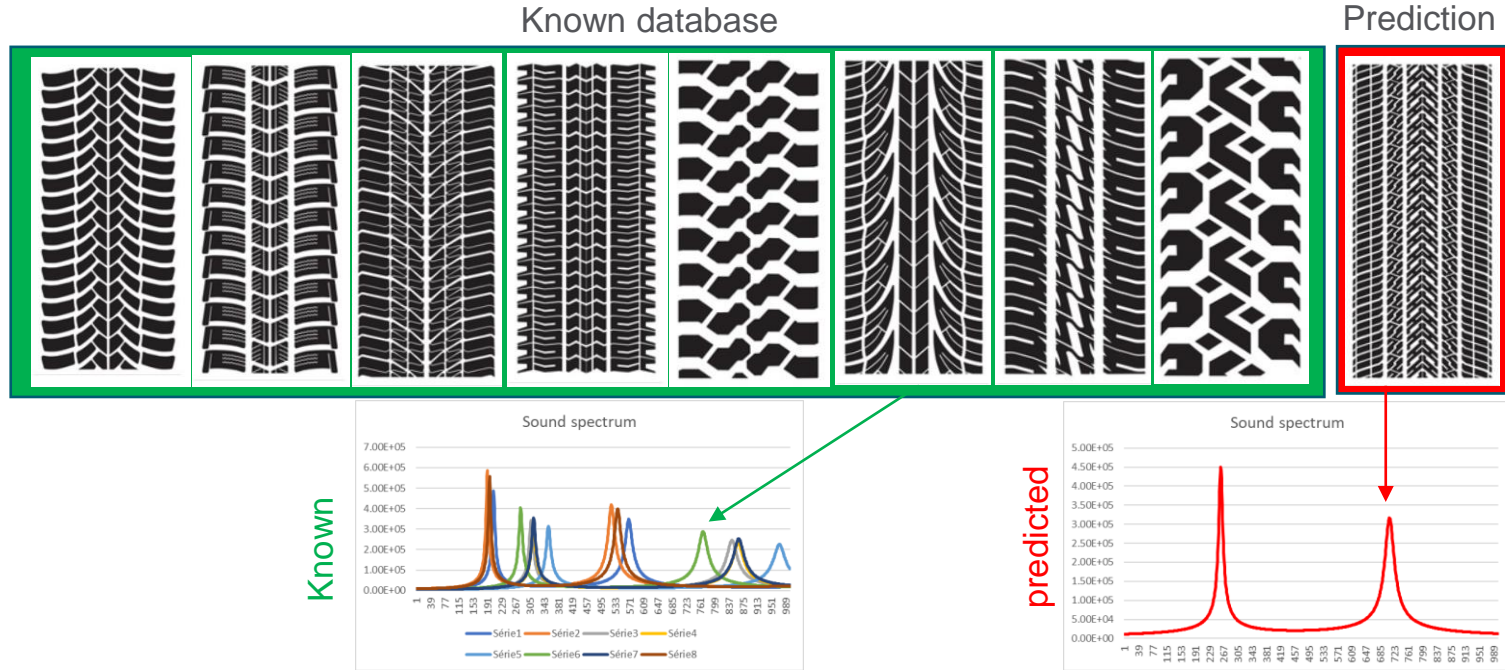


CFD



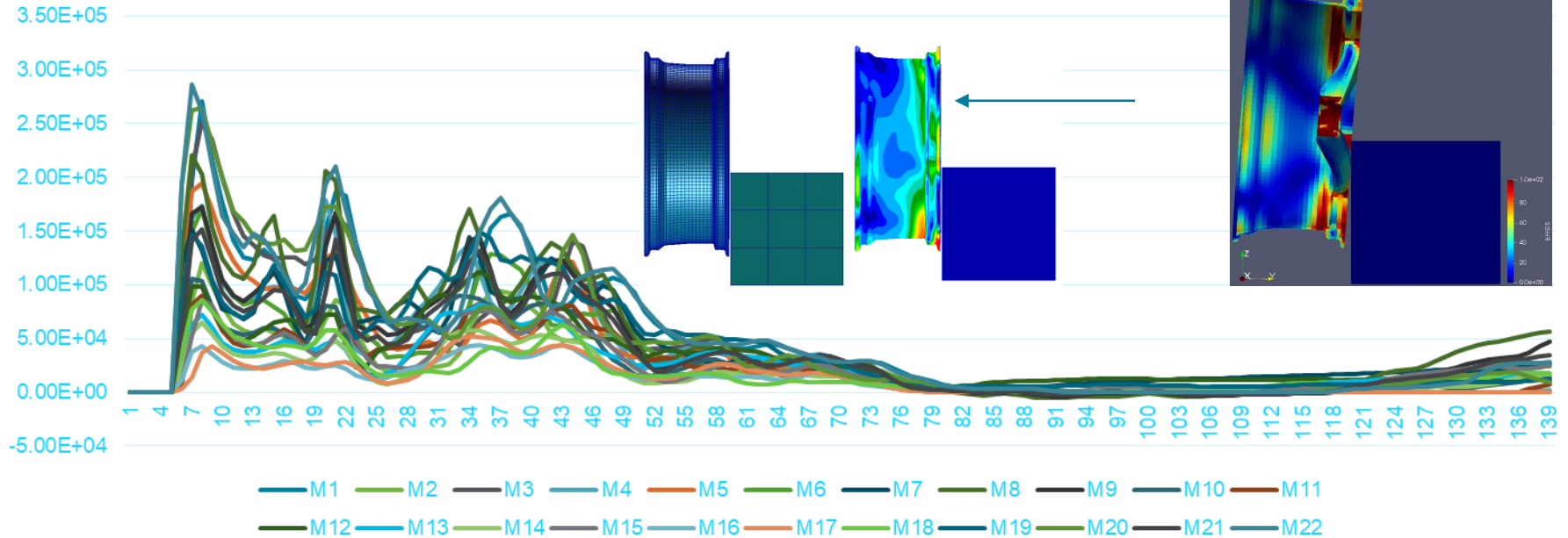
Neighbour = 2, pow = 0.0

Courtesy of Cradle (scStream)





Impact force (22 cases)

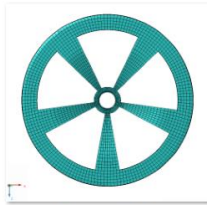




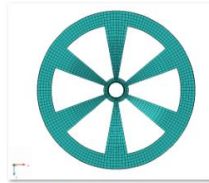


## X\_parameters

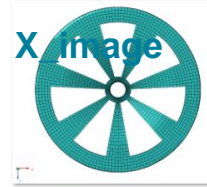
	Initial Velocity	Number of branches
M1	9	9
M2	4	9
M3	9	10
M4	8	8
M5	7	10
M6	6	9
M7	10	6
M8	10	7
M9	8	7
M10	5	8
M11	6	6
M12	7	5
M13	5	5
M14	3	8
M15	4	7
M16	3	6
M17	3	5
M18	3	10
M19	10	5
M20	10	10
M21	7	7
M22	10	9



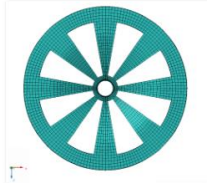
5\_Branches\_APEX.PNG



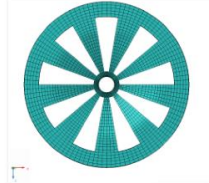
6\_Branches\_APEX.PNG



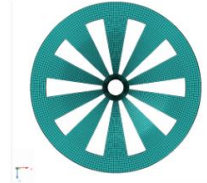
7\_Branches\_APEX.PNG



8\_Branches\_APEX.PNG

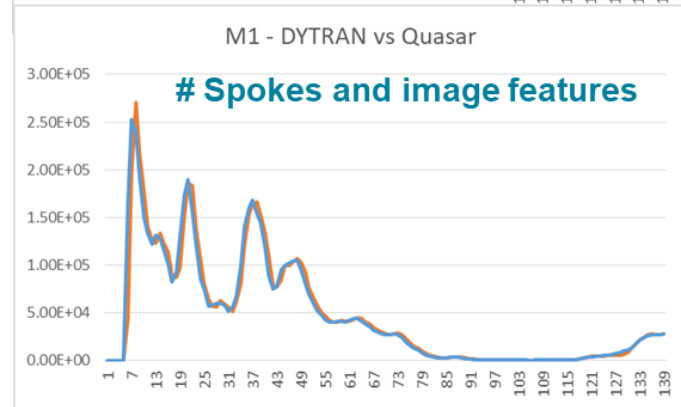
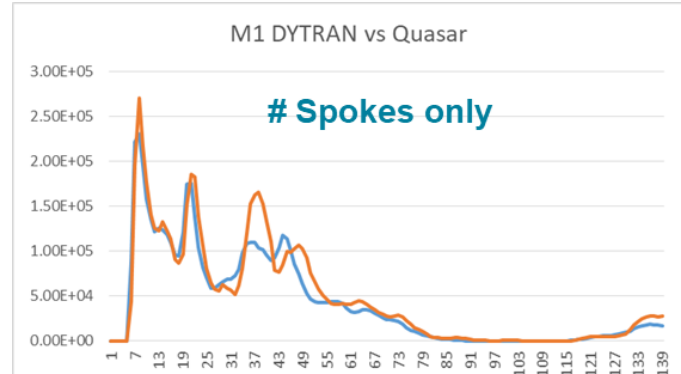


9\_Branches\_APEX.PNG



10\_Branches\_APEX.PNG

X\_image



# Full vehicle Passenger safety

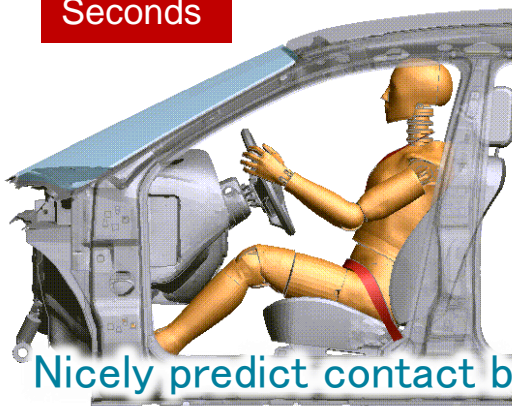
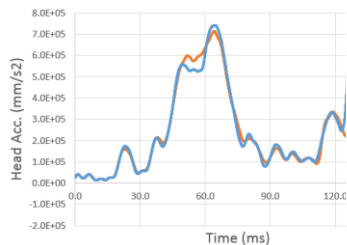


**ODYSSEE**

Explore new industrial horizons

Odyssee

Seconds



Nicely predict contact behavior between airbag and dummy

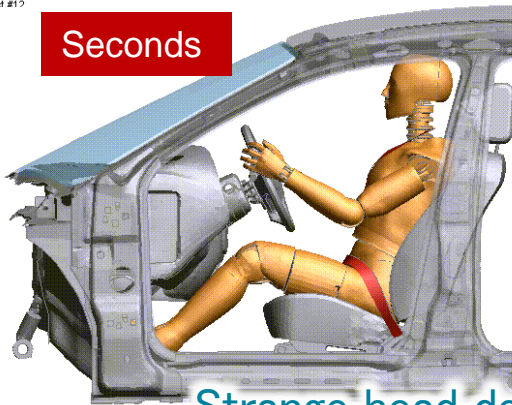
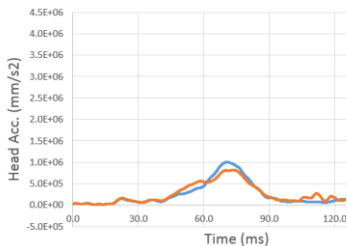
LS-DYNA

Hours x



Odyssee

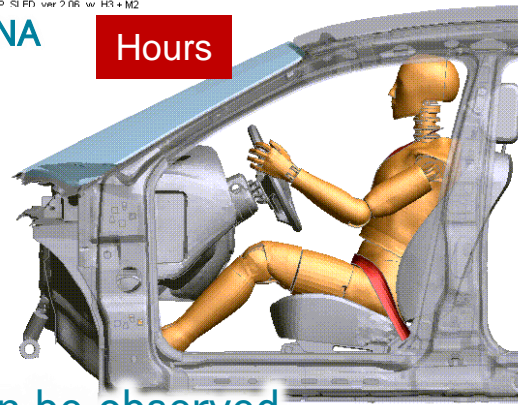
Seconds



Strange head deformation can be observed

LS-DYNA

Hours



Courtesy of JSOL

# Human biomechanics

	Training data		Computation time		
	DoF	# of data	Learning	Prediction(25steps)	Dyna (16 cores)
Arm	3	13	2m 6s	<b>1m 15s</b>	<b>9 h 40 m 56 s</b>
Leg	4	20	2m 54s	<b>2m 27s</b>	<b>11 h 7 m 42 s</b>

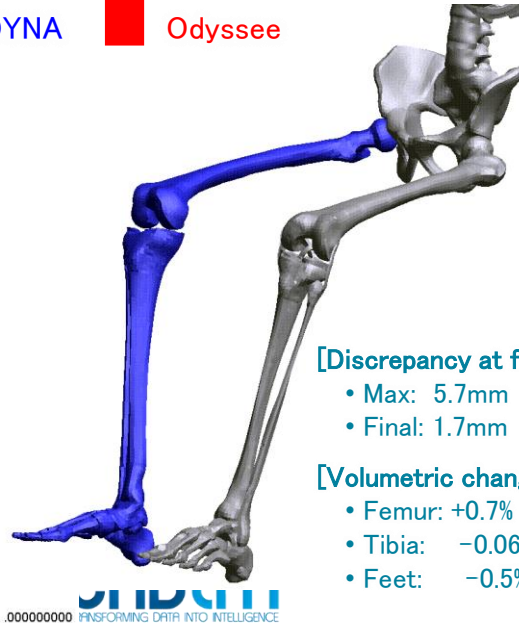
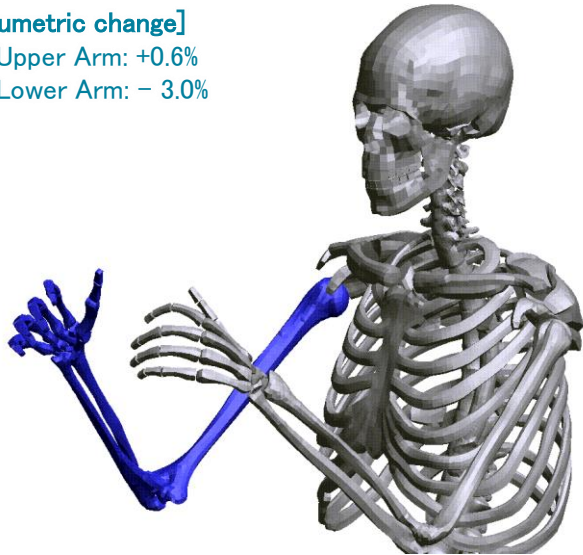
## [Discrepancy at fingertips]

- Max: 4.3mm
- Final:3.0mm

## [Volumetric change]

- Upper Arm: +0.6%
- Lower Arm: - 3.0%

 LS-DYNA  Odyssee



## [Discrepancy at fingertips]

- Max: 5.7mm
- Final: 1.7mm

## [Volumetric change]

- Femur: +0.7%
- Tibia: -0.06%
- Feet: -0.5%

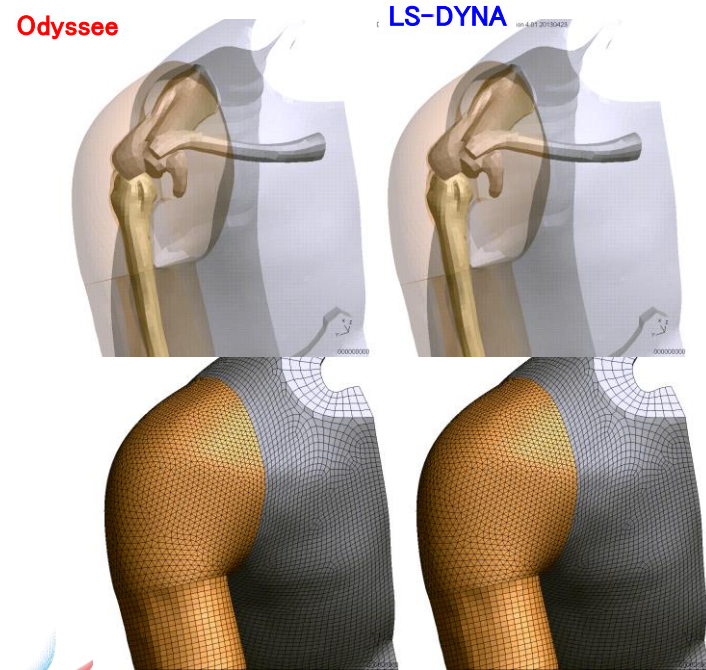
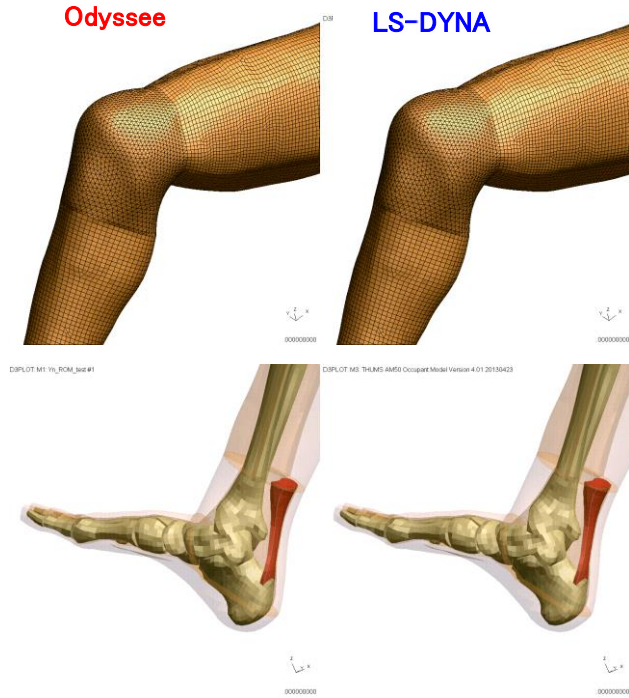
Courtesy of TOYOTA/JSOL

# Human biomechanics



**ODYSSEE**

Explore new industrial horizons

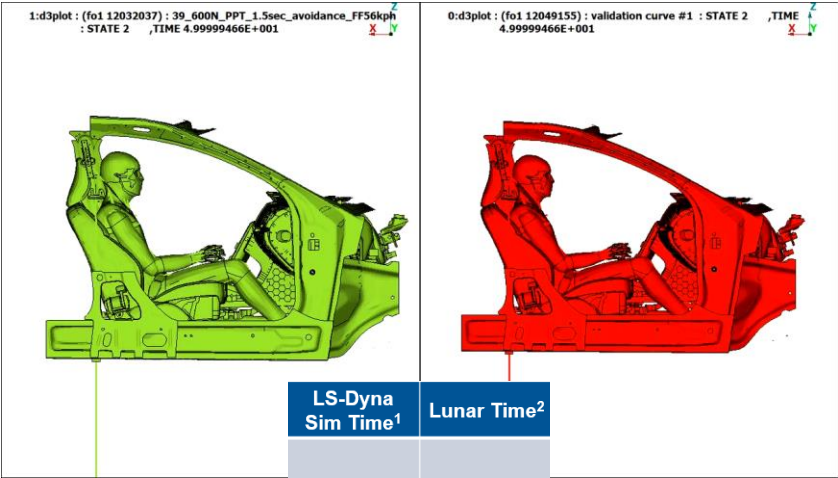
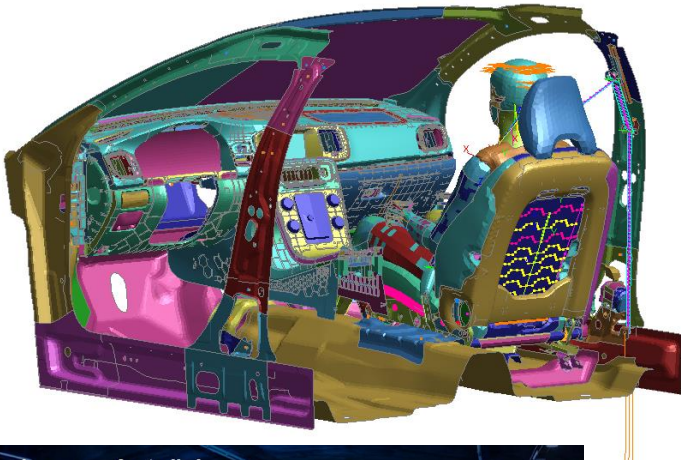


Courtesy of TOYOTA/JSOL

Odyssee: 1 min (1 core) vs. LS-DYNA: 10 hours (16 cores) to



# Pre-crash avoidance manoeuvre and crash



LS-Dyna Sim Time <sup>1</sup>	Lunar Time <sup>2</sup>
47h 35min	1h 32min

## 6 Runs only

1. Simulation carried out in Autoliv Research Cluster – PBS Nodes – Intel Xeon E5 (32 cores)
2. Calculations on company HP Zbook laptop (Intel Core i7 – 4910MQ – 4 cores)

Each year, Autoliv's products save over 30,000 lives

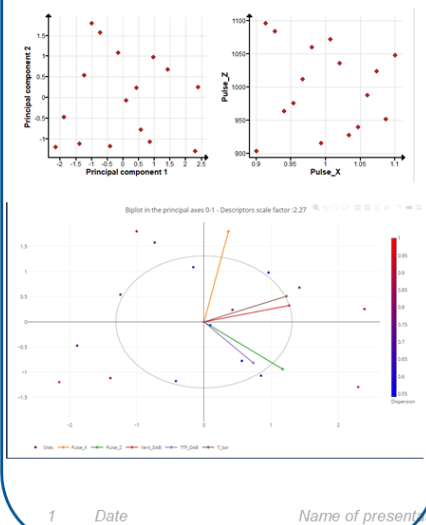
autoliv.com

“At AUTOLIV we are concerned with safety of real humans (and not only dummies). This requires a yet challenging computing effort for evaluation of our safety solutions. CADLM's ODYSSEE.Lunar software is a real breakthrough and provides a very promising perspectives in order to reduce drastically computing time and optimize our designs” Bengt Pipkorn (Director Simulation and Active Structures, Autoliv Research)

# Restraint system robustness Coupling CADLM / DynaS+ software

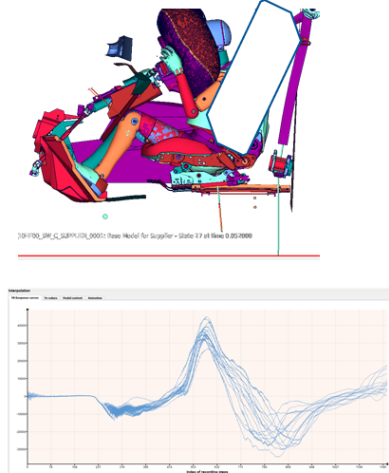
D.O.E

Lunar: Latin Hypercube



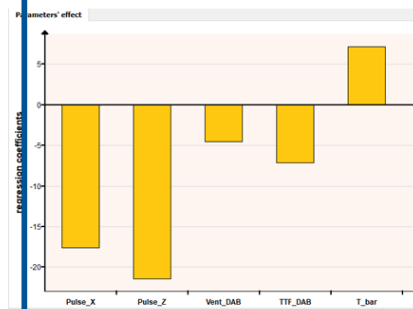
LSDYNA  
Simulations

Lunar : LSDYNA Model  
parsing for input/output



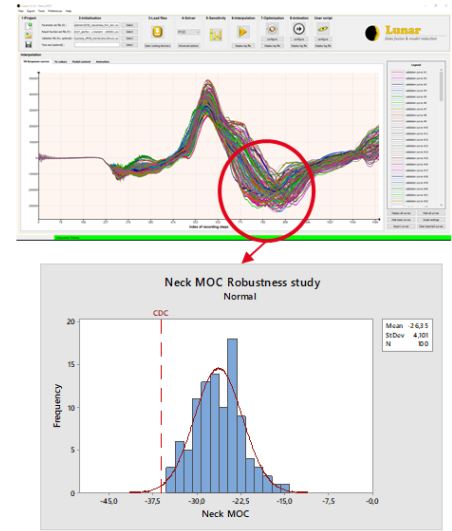
ROM

Lunar : Reduction Of  
Model



100 Monte Carlo runs

Lunar : Interpolation



**CPU times reduced by 86%**  
**Time of study : 5 days instead of 1 month**

**Autoliv**

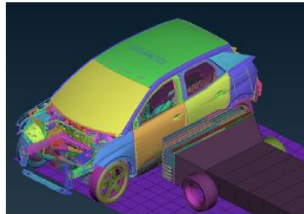
Courtesy of Autoliv France

## Side crashworthiness

### Side crashworthiness performance sizing



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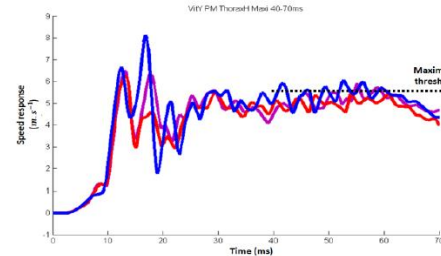


**New Peugeot 3008 side crash**  
Euro NCAP test and FE model

### Application of the NI-POD approach

- Test-case: Peugeot 3008 FE model
- Implementation: SVD done in Matlab and surrogate models calculated with the 'DiceKriging' R package.
- Tests done:
  - Prediction of various responses: displacements, velocities, forces...
  - Test of various surrogate models (GP, RBF...)
  - Various databases
  - Calculation of the prediction error

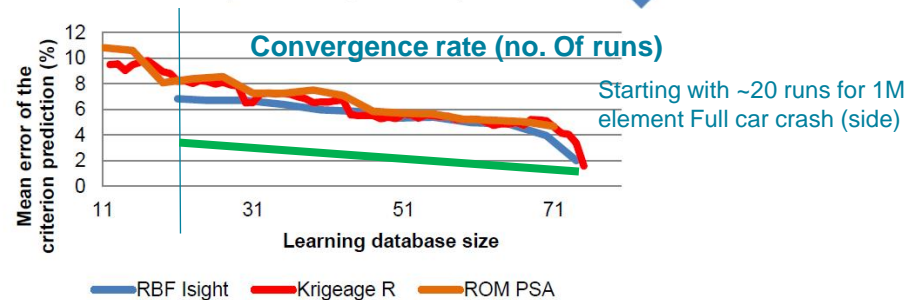
## Collaboration PSA/CADLM



- 2017
  - Scalars prediction only ☺
- 2018
  - Curves predictors
  - Deployment in 2019
- 2019
  - 3D/4D fields predictors
  - Deployment in 2020

### Criteria:

- Nodal displacements, velocities, stresses...



## Development of Prediction Method by Reduced Model for Structural Deformations in Frontal Impact

Hota Hashimoto, Hiroaki Onodera, Yasuo Yamane, Tsuyoshi Yasuki, Development of Prediction Method by Reduced Model for Structural Deformations in Frontal Impact, JSAE Paper Number: 20180154, Oct, 2018 Issued No.135-18



Hota Hashimoto

Hiroaki Onodera

Yasuo Yamamae

Tsuyoshi Yasuki



Fig. 2 Car crash FE model

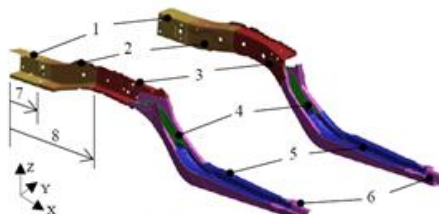


Fig. 3 Design parameters

Table 1 Maximum and minimum values of thickness and length of difference thickness for design parameters

Part	Design parameter	Min.	Max.
1	Thickness	1.1 mm	2.5 mm
2	Thickness	1.4 mm	2.8 mm
3	Thickness	1.9 mm	3.3 mm
4	Thickness	1.3 mm	2.7 mm
5	Thickness	0.7 mm	2.1 mm
6	Thickness	0.9 mm	2.3 mm
7	Length of difference thickness	100 mm	300 mm
8	Length of difference thickness	400 mm	640 mm

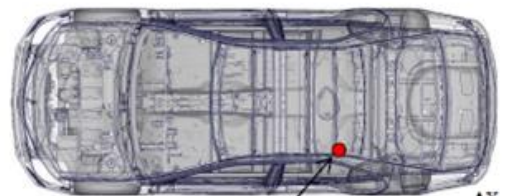


Fig. 4 Measurement node for vehicle acceleration and displacement

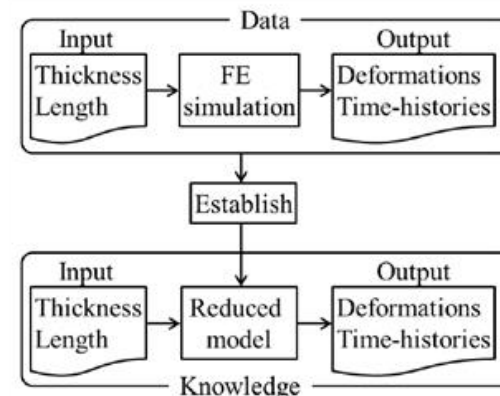


Fig. 1 Prediction method for deformations and time-histories



## Correlation

## Prediction

## Clustering

Hota Has himoto, Hiroaki Onodera, Yasuo Yamane, Tsuyoshi Yasuli, Development of Prediction Method by Reduced Model for Structural Deformations in Frontal Impact, JSAE Paper Number: 20180154, Oct, 2018 Issued No.135-18

Reduced model による最大車両加速度と最大移動量の予測精度を図 10, 11 に示す。最大加速度の平均誤差は 6%, 最大移動量の平均誤差は 1% だった。

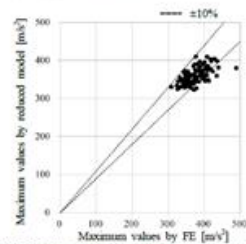


Fig. 10 Validation of maximum values of vehicle acceleration

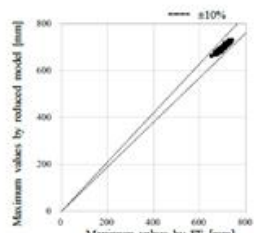


Fig. 11 Validation maximum values of vehicle displacement

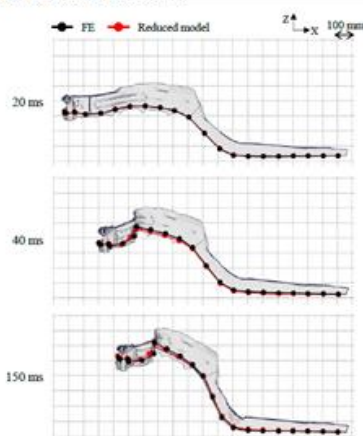


Fig. 13 Deformation of front side rail in Case 15

- (1) 熊口純一ほか: Reduced Model による SUV 衝突時のセンタービーター変形モードの予測, 自動車技術会論文集, Vol.49, No.2, p359-364(2018)
- (2) 小野寺啓祥ほか: 衝突乗員傷害低減のための Reduced Model の開発, 自動車技術会 2017 年秋大会 学術講演会 講演予稿集, No. 159, p1277-1282, 20176235(2017)
- (3) Kambiz kayvanfar, et al.: Model Reduction Techniques for LS-DYNA ALE and Crash Applications, Proceedings of 10th European LS-DYNA Conference(2015)
- (4) NCAC, Toyota Camry Finite Element Model Version 1, Technical Report, The George Washington University, 2014

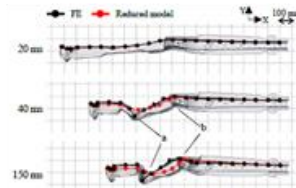


Fig. 14 Deformation of front side rail in Case 61

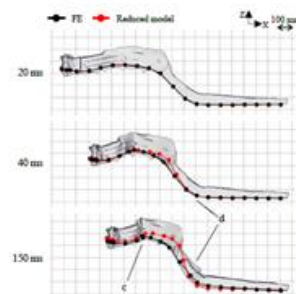


Fig. 15 Deformation of front side rail in Case 61

3.3. 変形モードのクラスタリング  
Reduced model による変形モードが FE 結果と異なるケースがあった (図 14, 15)。変形モードが複数存在するための予測

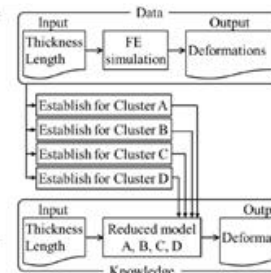


Fig. 16 Prediction method by clustering deformations

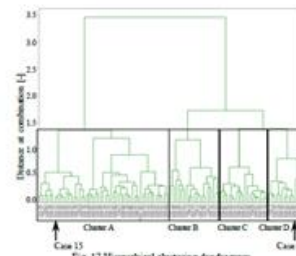


Fig. 17 Hierarchical clustering dendrogram



**TOYOTA**

Hota Has himoto, Hiroaki Onodera, Yasuo Yamane, Tsuyoshi Yasuli, Development of Prediction Method by Reduced Model for Structural Deformations in Frontal Impact, JSAE Paper Number: 20180154, Oct, 2018 Issued No.135-18

## Development of Reduced Model for Aerodynamic Drag and Lift

Mashio Taniguchi, Junichi Inokuchi, Yasuo Yamamae, Development of Reduced Model for Aerodynamic Drag and Lift, JSAE Paper Number: 20186092, Oct, 2018 Issued No.121-18

Mashio Taniguchi

Junichi Inokuchi

Yasuo Yamamae

Hiroshi Tanaka

Tsuyoshi Yasuki

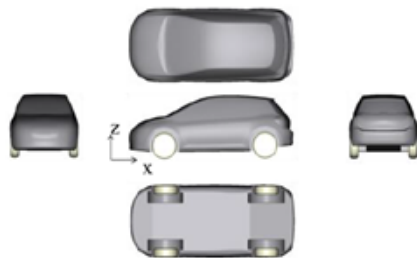


Fig. 2 Vehicle model (Base)

Table 2 Design variables and range

Design Variable	Base	Minimum	Maximum
A1 [deg]	14.0	14.0	24.0
H1 [mm]	144.7	144.1	194.0
H2 [mm]	2.6	2.6	129.7
H3 [mm]	0.0	-16.5	278.9
L1 [mm]	819.6	821.7	1336.7
L2 [mm]	997.4	997.4	1196.8
R1 [mm]	0.0	-771.8	777.7
W1 [mm]	45.5	45.5	199.5
W2 [mm]	29.5	25.3	109.8

ベース形状での車両中央断面 (y=0mm) における圧力分布を図 5 に示す。ループ後端からせん断渦が放出され、時間経過とともに車両後方へ移動しており、患部の研究における流れ場と同様であった。

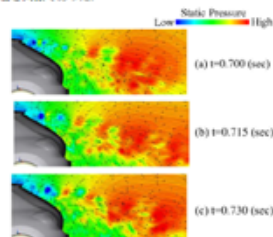


Fig. 5 Pressure Distribution at y=0 Plane (Base)

Reduced Model に新たな変数を入力して,  $C_D$ ,  $C_L$  を算出する.

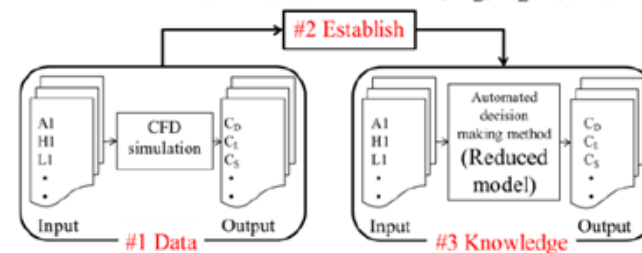
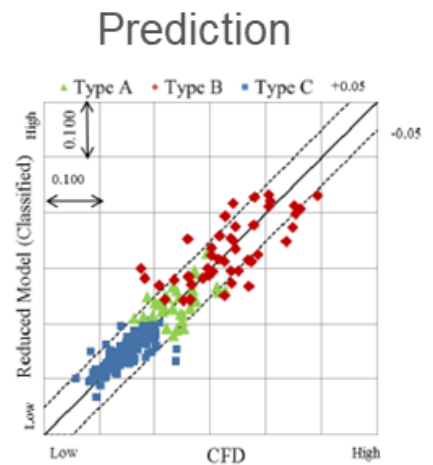
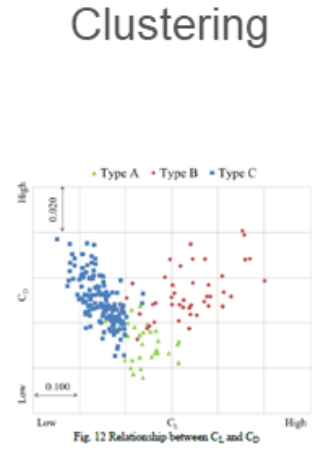
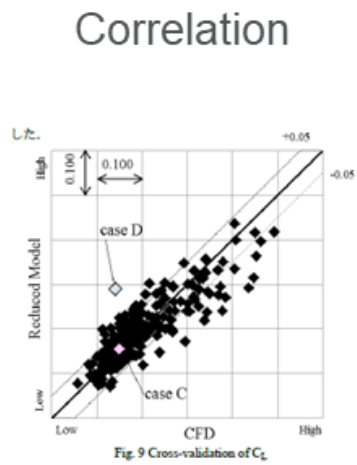
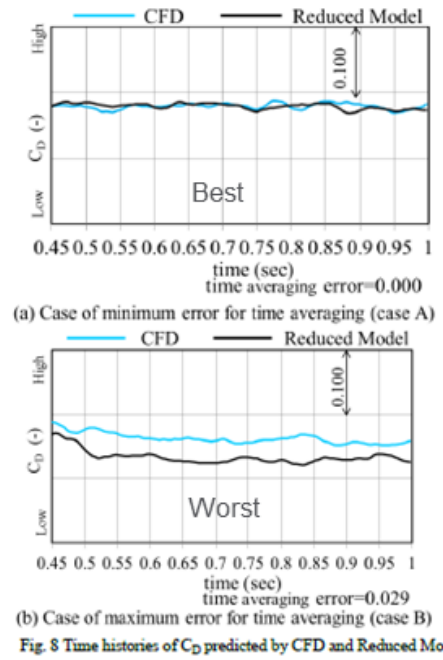


Fig. 1 Automated decision making method





Mashio Taniguchi; Junichi Inokuchi; Yasuo, Development of Reduced Model for Aerodynamic Drag and Lift, JSAE Paper Number: 20186092, Oct, 2018 Issued No.121-18



- (5) 忍津雅也, 山村淳, 田中博, 安木剛, 高山裕, 山出吉伸, 鈴木康方, 加藤千幸: 2Box 車の空気抵抗低減に関する研究, 自動車技術会論文集, Vol.44, No.5, p.1287-1294 (2013)
- (6) Optimal Decision Support System for Engineering and Expertise (<http://odyssee.cadlm.com>)
- (7) R. Kotapati, A. Keating, S. Kandasamy, B. Duncan, R. Shock and H. Chen : The Lattice-Boltzmann-VLES Method for Automotive Fluid Dynamics Simulation, a Review, SAE Paper No.2009-29-057, p.2-3 (2009)



# Try it ...

## Quasar Online

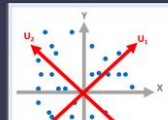
SLECORRE

🏠 **UPLOAD** VISUALIZATION DATA MINING MODEL REDUCTION FORECASTING OPERATION D.O.E SCRIPT TIPS & TRICKS

Modelling shortcut



K-means clustering



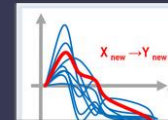
PCA



Regression



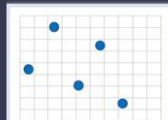
Dendrogram plot



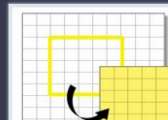
Reduced Order Model



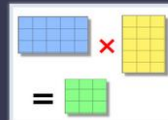
Forecasting



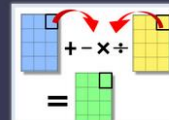
Design of experiment



Matrix extraction



Matrix multiplication



Element-wise operation

Demo videos



Quasar Online introduction



Quasar Online CSV convention



CSV files for Model Reduction tab